



# Storing Petroleum Liquids in Large Aboveground Storage Tanks

PEMY Consulting LLC

Philip Myers

**22nd Annual California CUPA Training Conference**

**THEME: "2020: PERFECTING OUR VISION"**

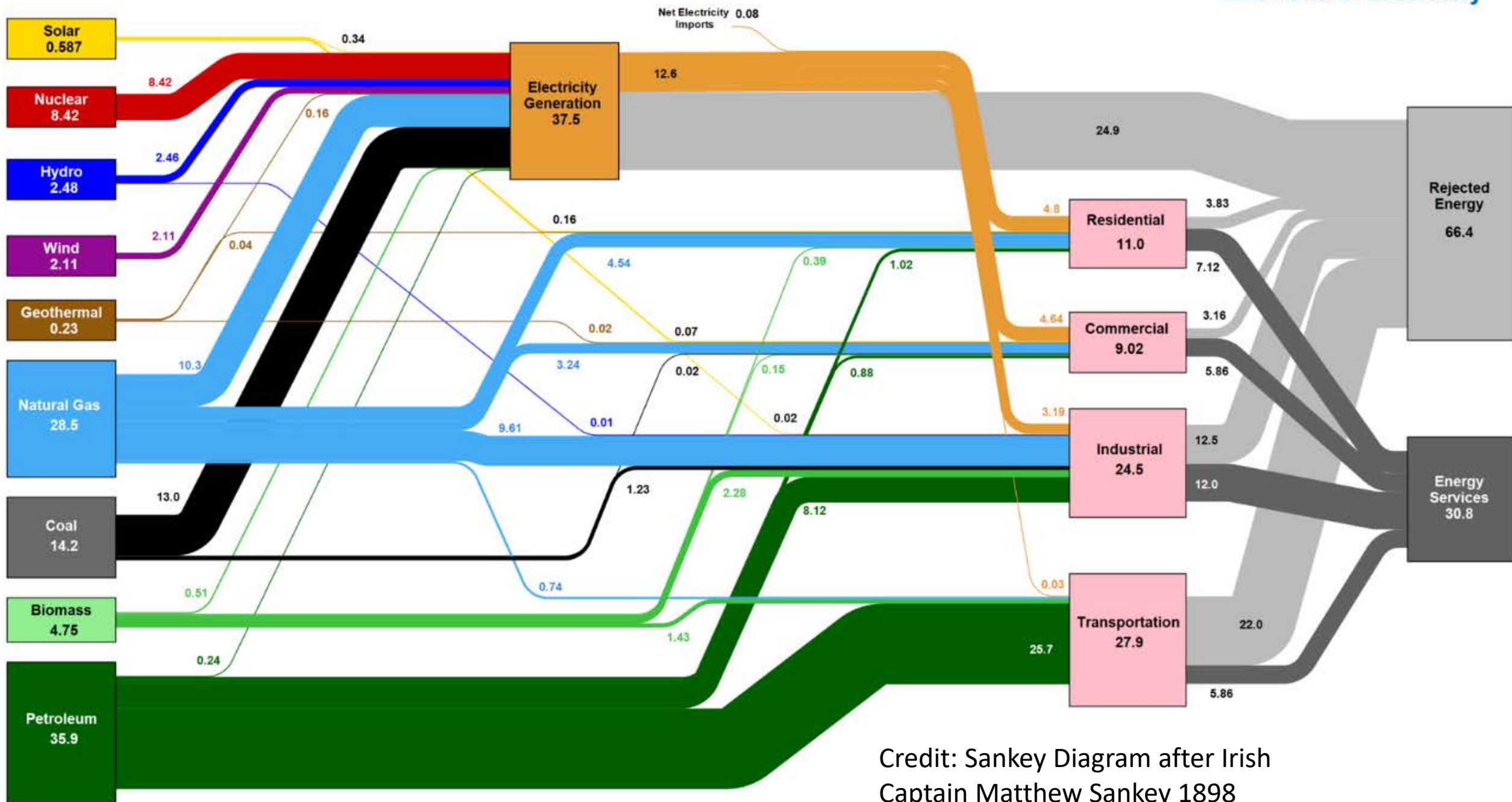
**DATES: Feb 3rd thru Feb 6th, 2020**

# U.S Energy Flow

- “Quad”  $10^{15}$  BTU, equivalent of 183,000,000 bbls oil
- World energy consumption was 524 quads in 2019
- U.S. energy consumption in 2016 was 97 quads
- eia.gov
- <https://www.eia.gov/state/maps.php>

Wow – we use 20%!

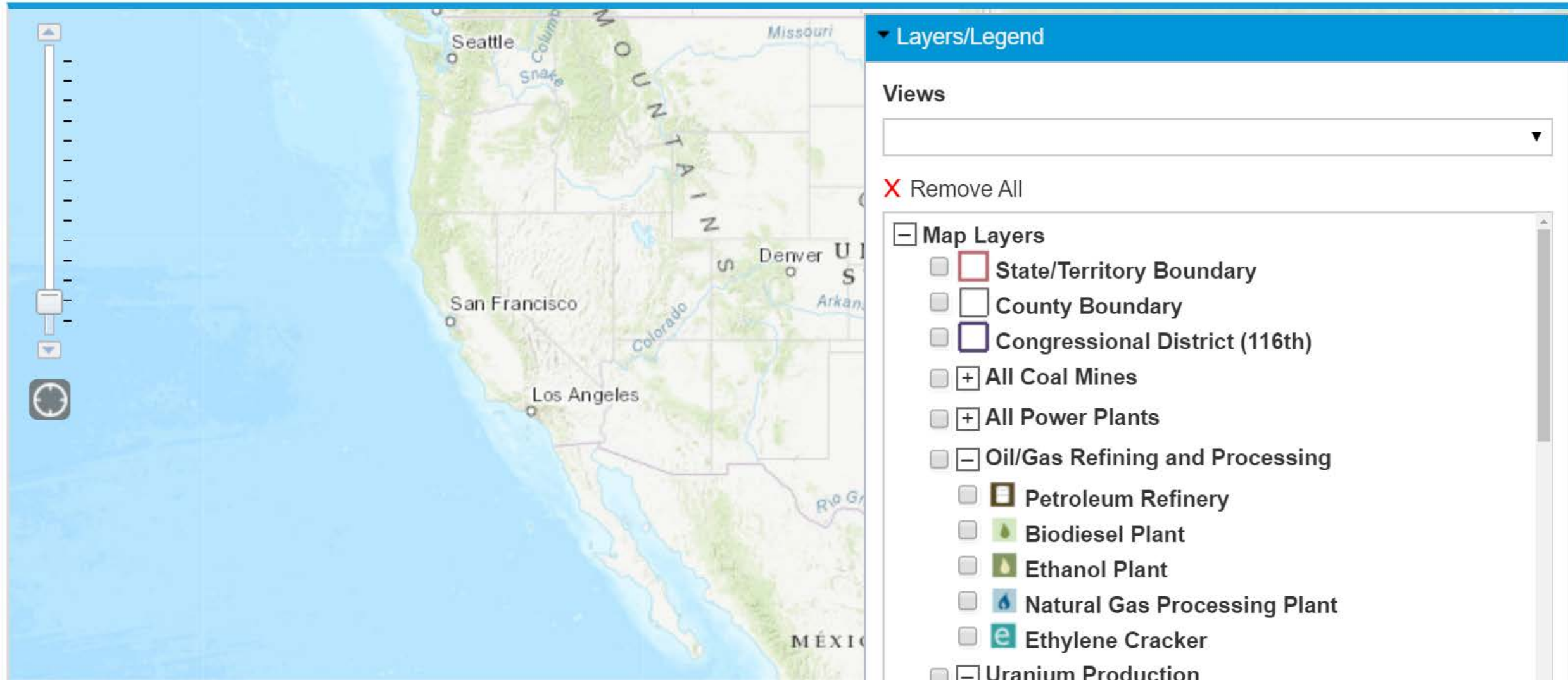
# Estimated U.S. Energy Consumption in 2016: 97.3 Quads



Credit: Sankey Diagram after Irish Captain Matthew Sankey 1898



# U.S. Energy Mapping System

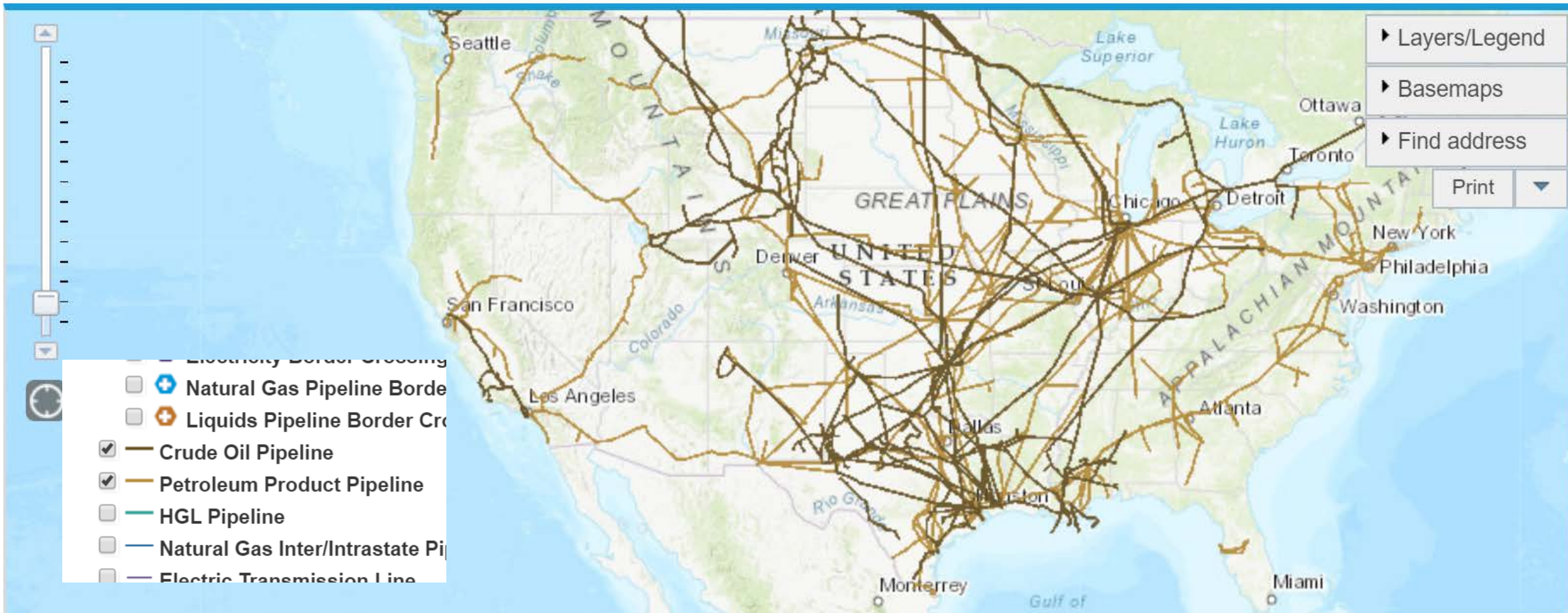


The screenshot displays the U.S. Energy Mapping System interface. On the left, a map of the western United States is shown, featuring major cities like Seattle, San Francisco, and Los Angeles, and geographical features like the Sierra Nevada mountains and the Colorado River. A vertical scale bar and navigation controls are visible on the left side of the map.

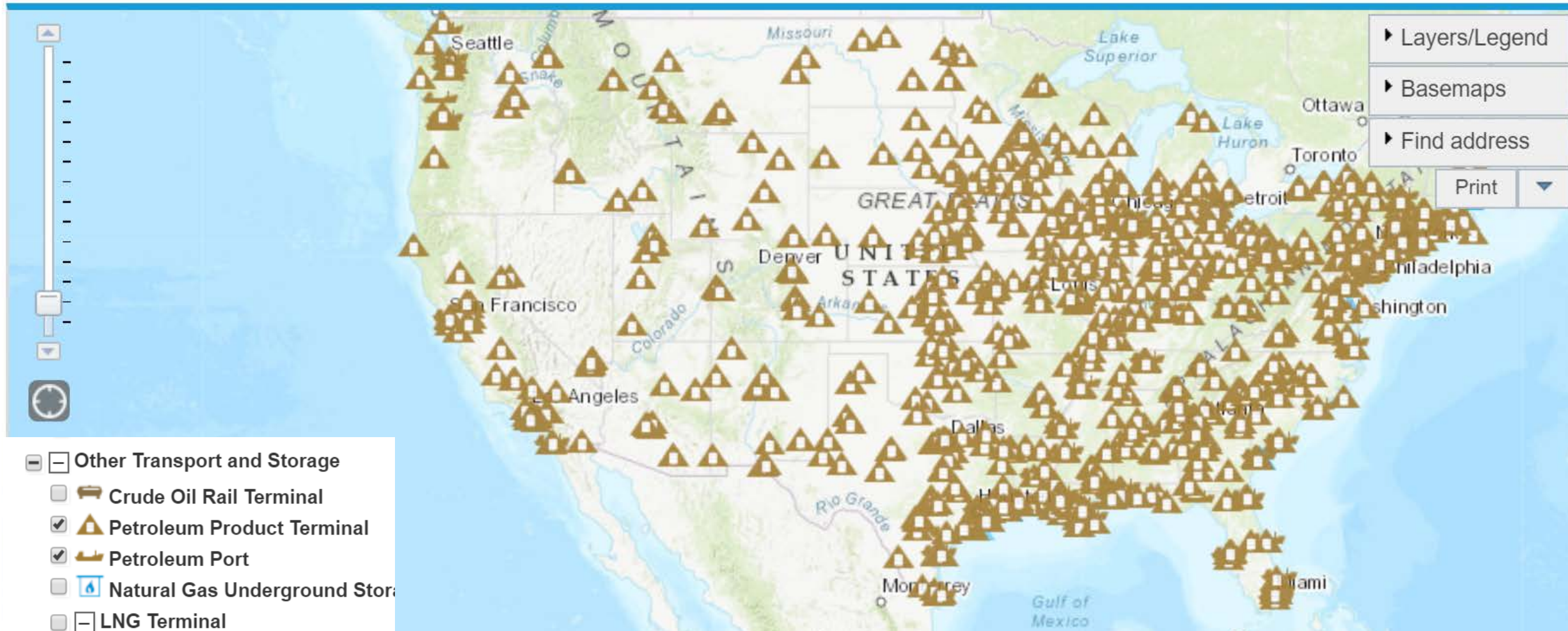
On the right, the "Layers/Legend" panel is open, showing a "Views" dropdown menu and a "Remove All" button. Below these, the "Map Layers" section lists various energy-related features with checkboxes and icons:

- State/Territory Boundary
- County Boundary
- Congressional District (116th)
- + All Coal Mines
- + All Power Plants
- Oil/Gas Refining and Processing
  - Petroleum Refinery
  - Biodiesel Plant
  - Ethanol Plant
  - Natural Gas Processing Plant
  - Ethylene Cracker
- Uranium Production

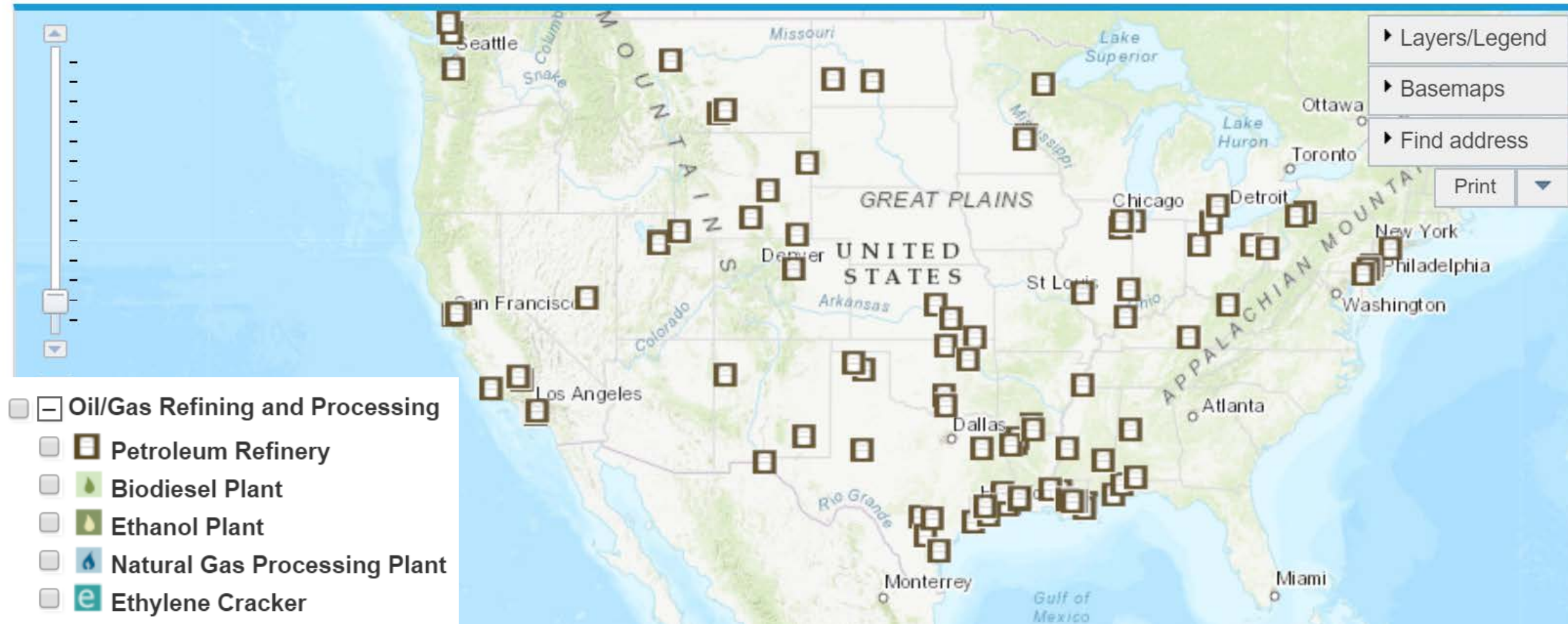
# U.S. Energy Mapping System



# U.S. Energy Mapping System



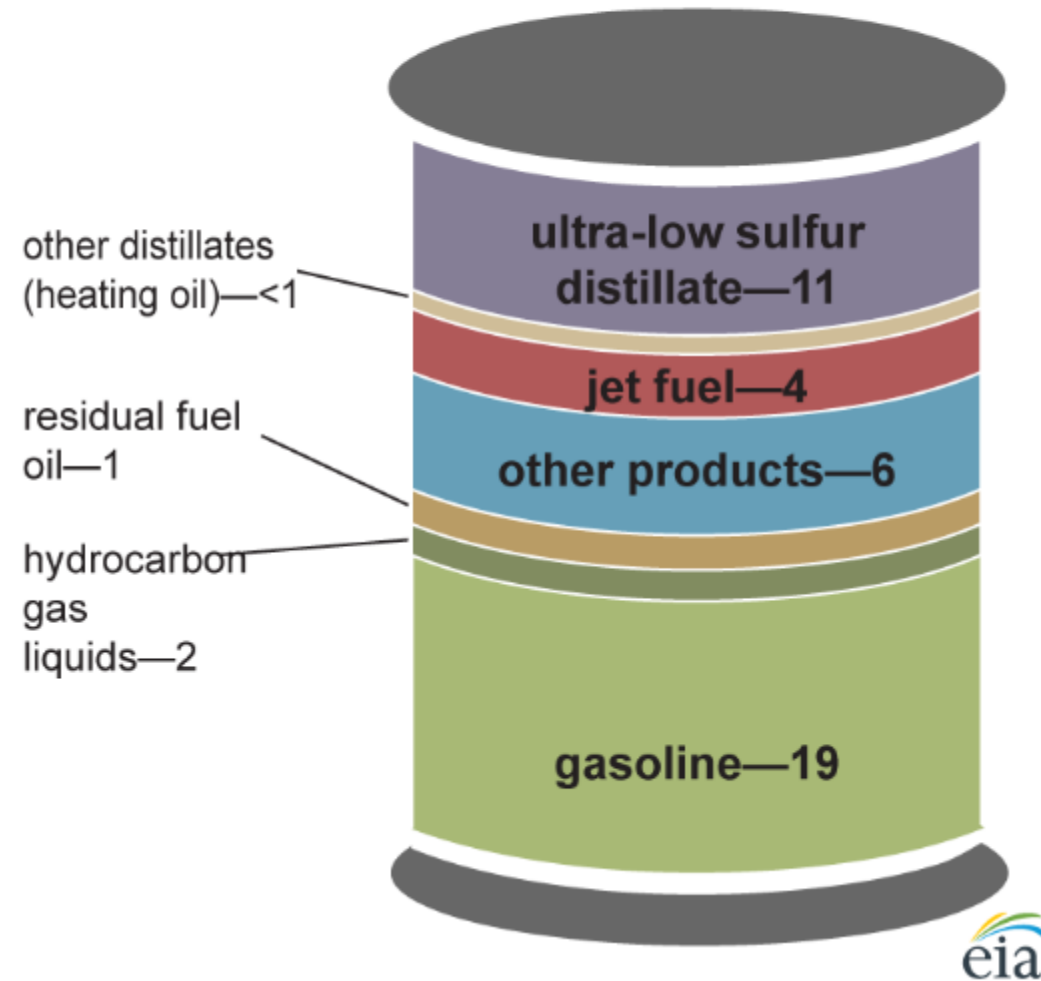
# U.S. Energy Mapping System





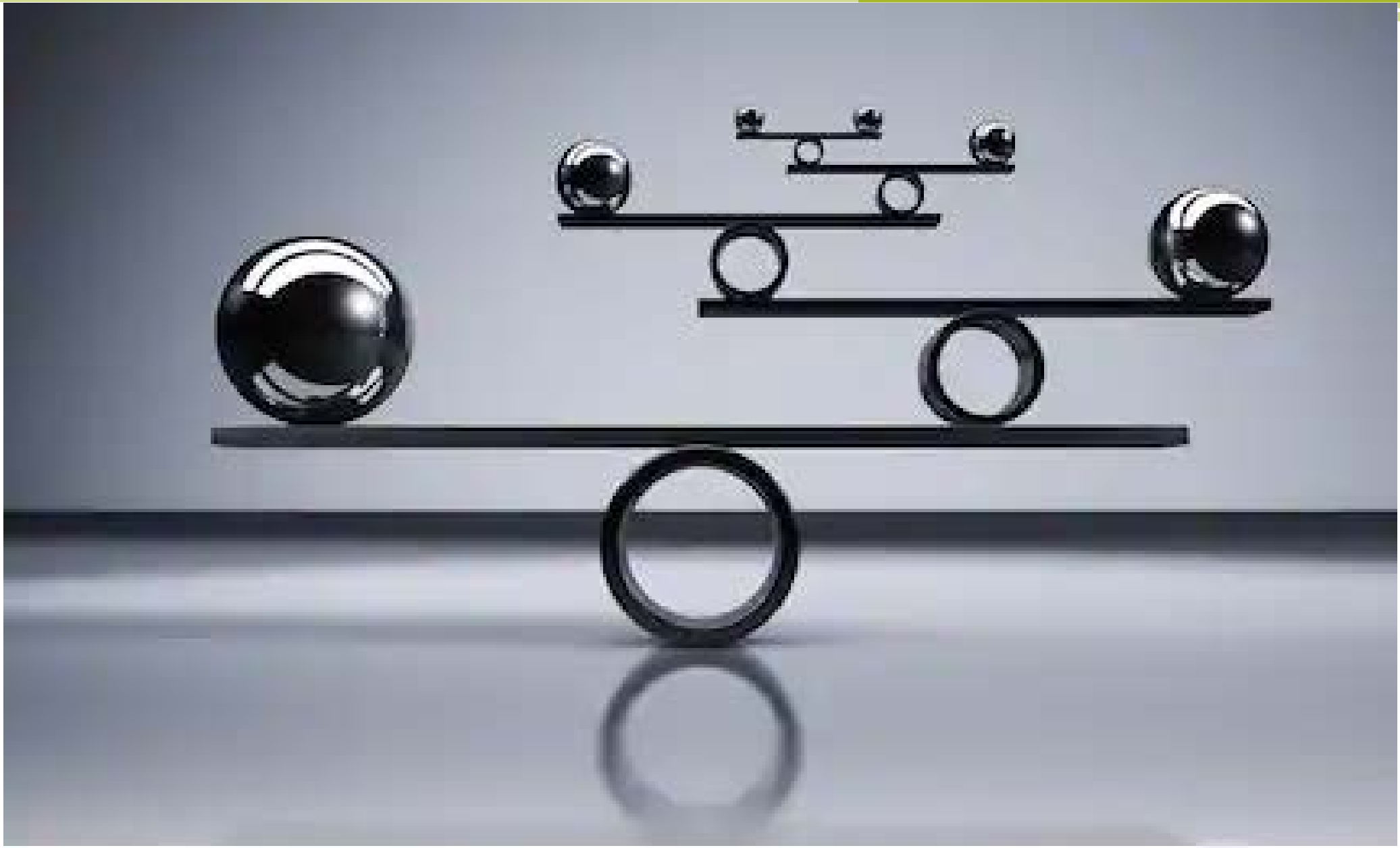
# Petroleum products made from a barrel of crude oil, 2018

gallons



About ½ barrel of crude becomes gasoline – the rest -

Solvents	Diesel fuel	Motor Oil	Bearing Grease
Ink	Floor Wax	Ballpoint Pens	Football Cleats
Upholstery	Sweaters	Boats	Insecticides
Bicycle Tires	Sports Car Bodies	Nail Polish	Fishing lures
Dresses	Tires	Golf Bags	Perfumes
Cassettes	Dishwasher parts	Tool Boxes	Shoe Polish
Motorcycle Helmet	Caulking	Petroleum Jelly	Transparent Tape
CD Player	Faucet Washers	Antiseptics	Clothesline
Curtains	Food Preservatives	Basketballs	Soap
Vitamin Capsules	Antihistamines	Purses	Shoes
Dashboards	Cortisone	Deodorant	Footballs
Putty	Dyes	Panty Hose	Refrigerant
Percolators	Life Jackets	Rubbing Alcohol	Linings
Skis	TV Cabinets	Shag Rugs	Electrician's Tape
Tool Racks	Car Battery Cases	Epoxy	Paint
Mops	Slacks	Insect Repellent	Oil Filters
Umbrellas	Yarn	Fertilizers	Hair Coloring
Roofing	Toilet Seats	Fishing Rods	Lipstick
Denture Adhesive	Linoleum	Ice Cube Trays	Synthetic Rubber
Speakers	Plastic Wood	Electric Blankets	Glycerin
Tannic Acids	Rubber Cement	Fishing Boats	Dice



# Valuing: Health, Safety, Environment

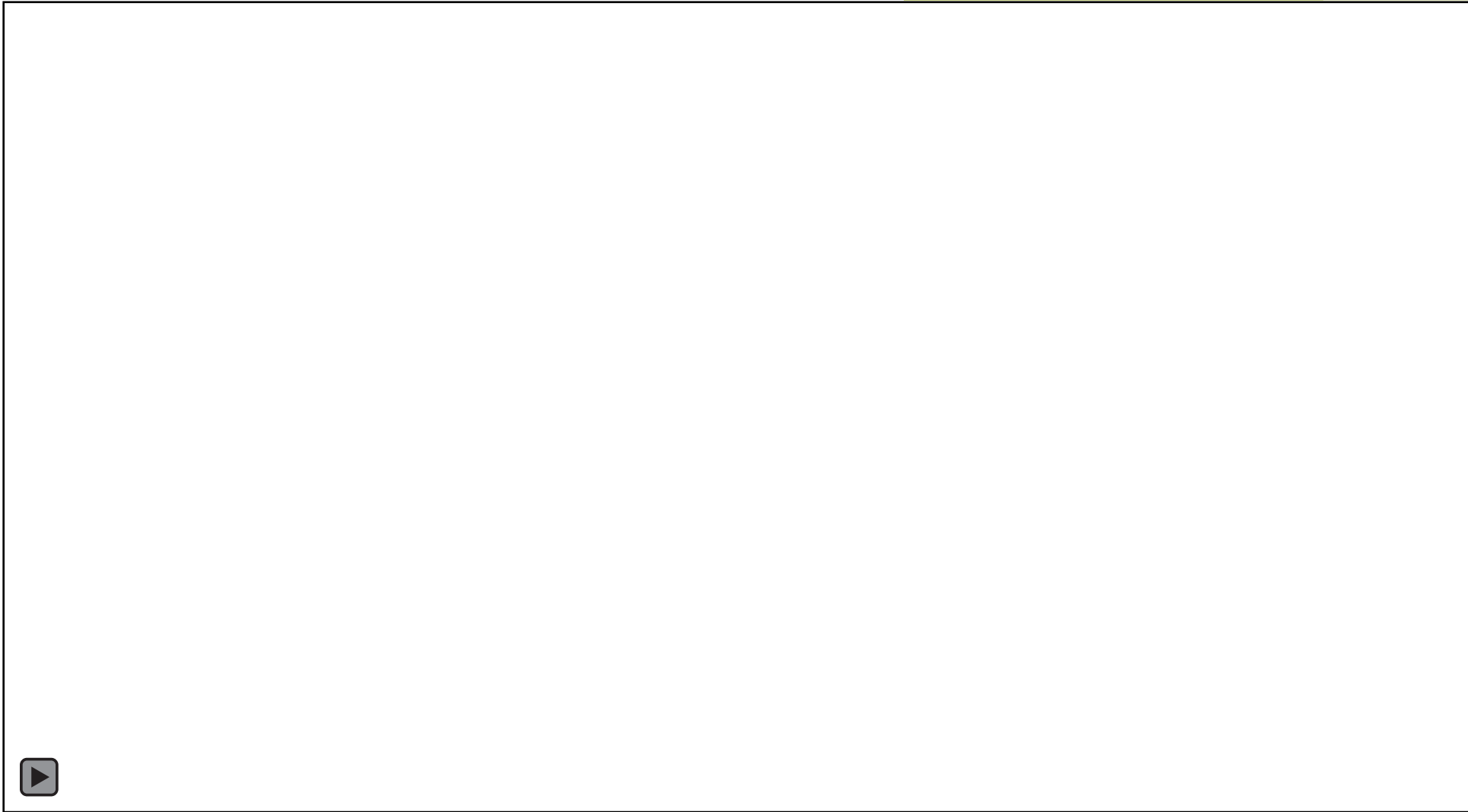




8 covered tanks (2019 Dorian)



4 covered tanks (2019 Dorian)













# What are impacts of these events?

- Business interruption
- Environment
- Reputation
- Good design
- Employee alignment
- Stay out of news
- Meaning of risk

**Should society tolerate these events?**



**Tanks Anyone?**



Molten Steel  
Storage - crucible

Are these tanks?



Acid  
Storage



Are These Tanks?







© Dan L. Perlman/EcoLibrary.org DP4505

Is this a tank?



# Fuel Tanks

















**But What About Oil Storage Tanks?**





Image courtesy of Nabucco Gas Pipeline International GmbH

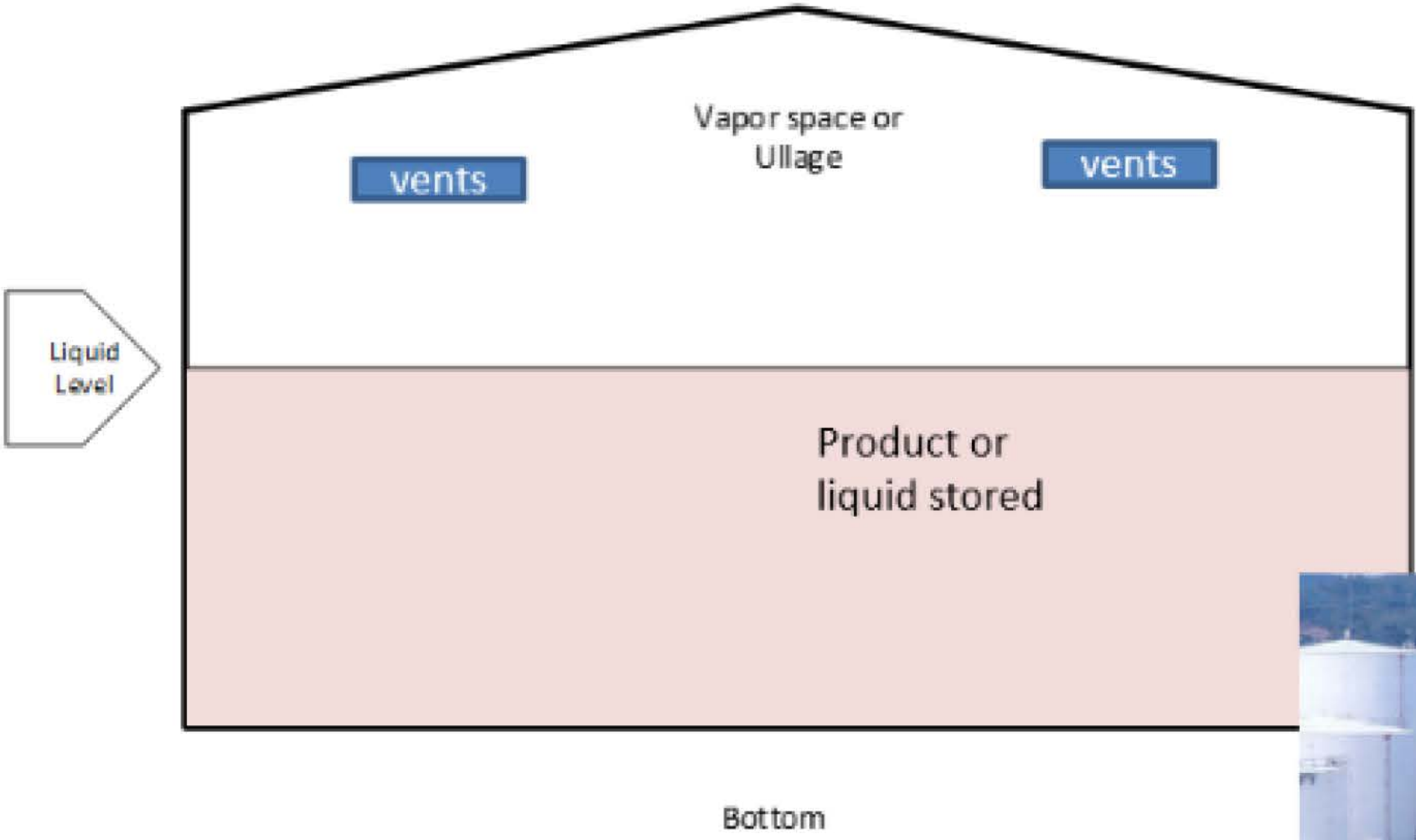








# Fixed Roof Tank (FRT)



Vent function not shown for clarity; can be opening in vapor space or a PV valve

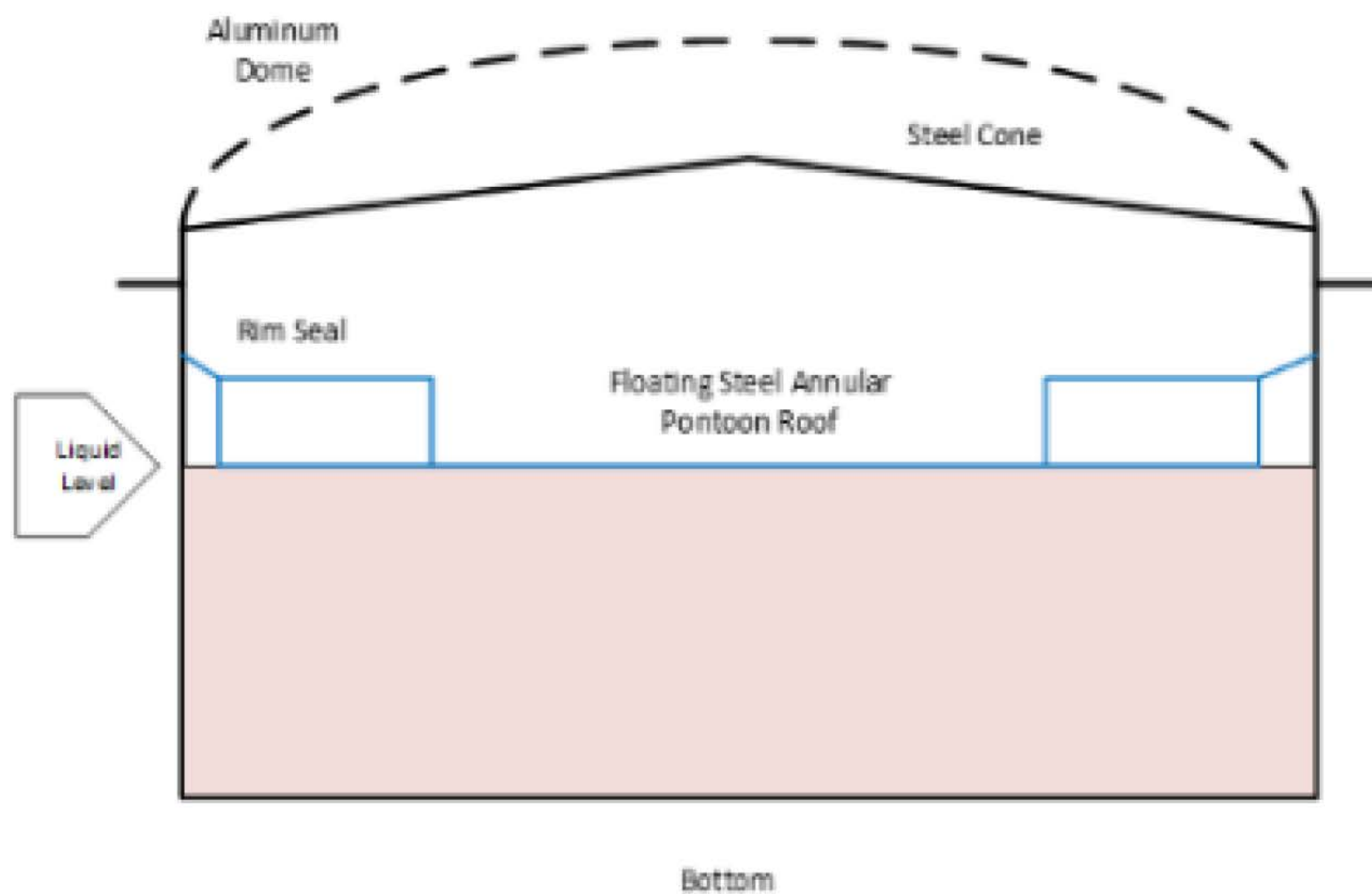


Figure 22 Fixed Roof Tank (FRT)





# Covered or Internal Floating Roof Tank



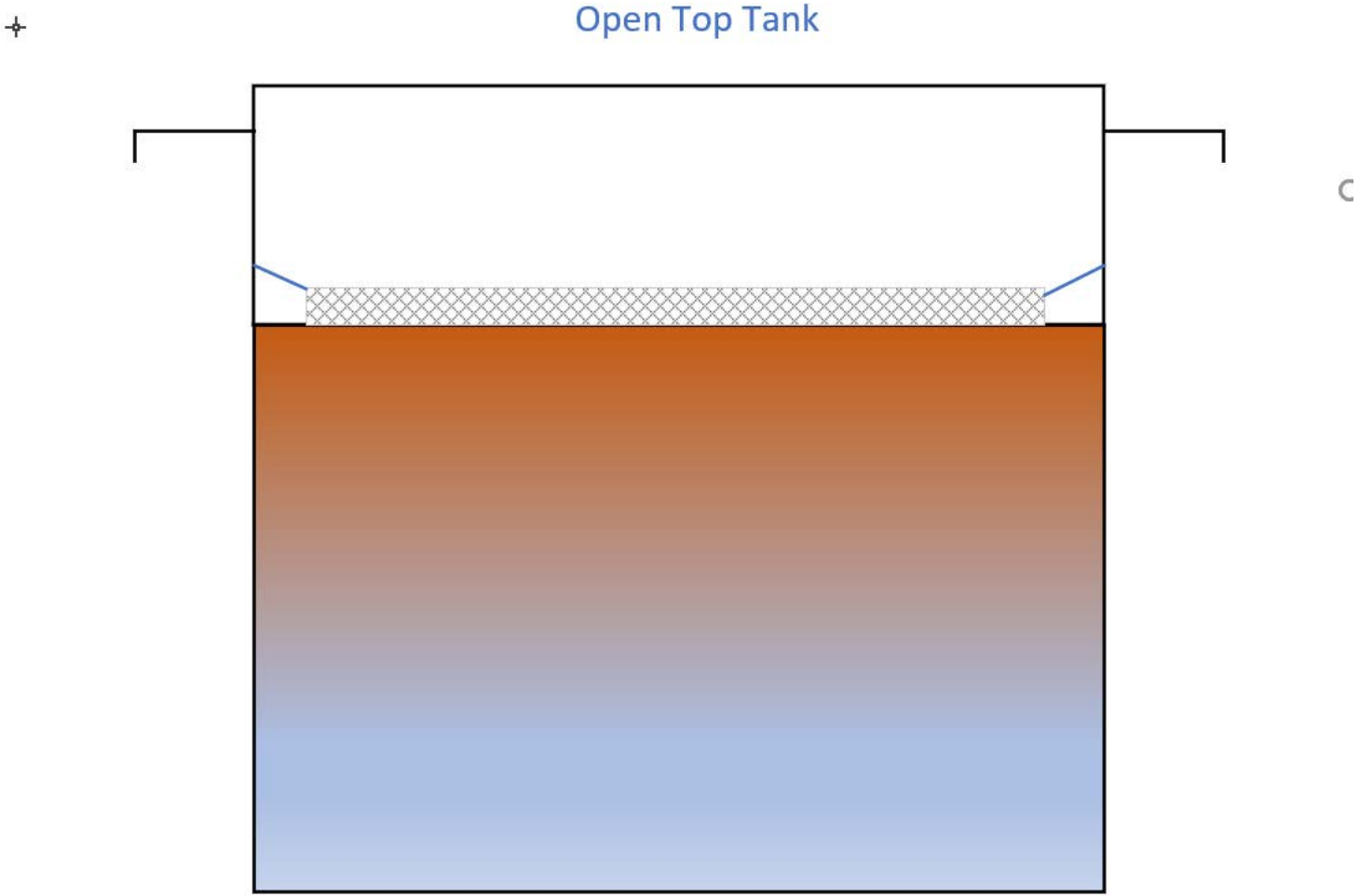
Open Top Wind Girder  
(Stiffener)

Shell

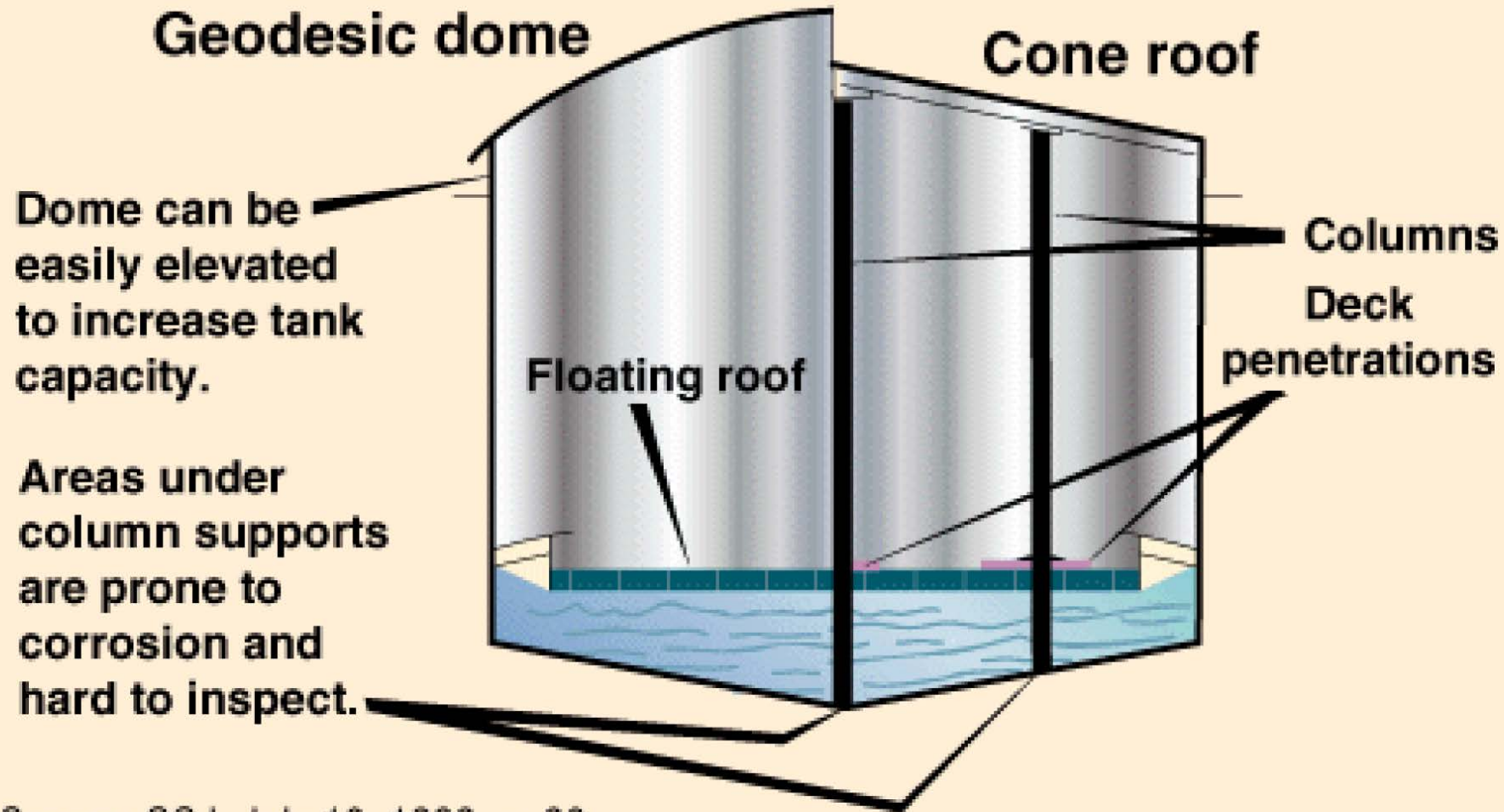


Figure 28: Internal floating Roof Tank (IFRT)

# External Floating Roof Tank



# GEODESIC DOME VS. CONE ROOF

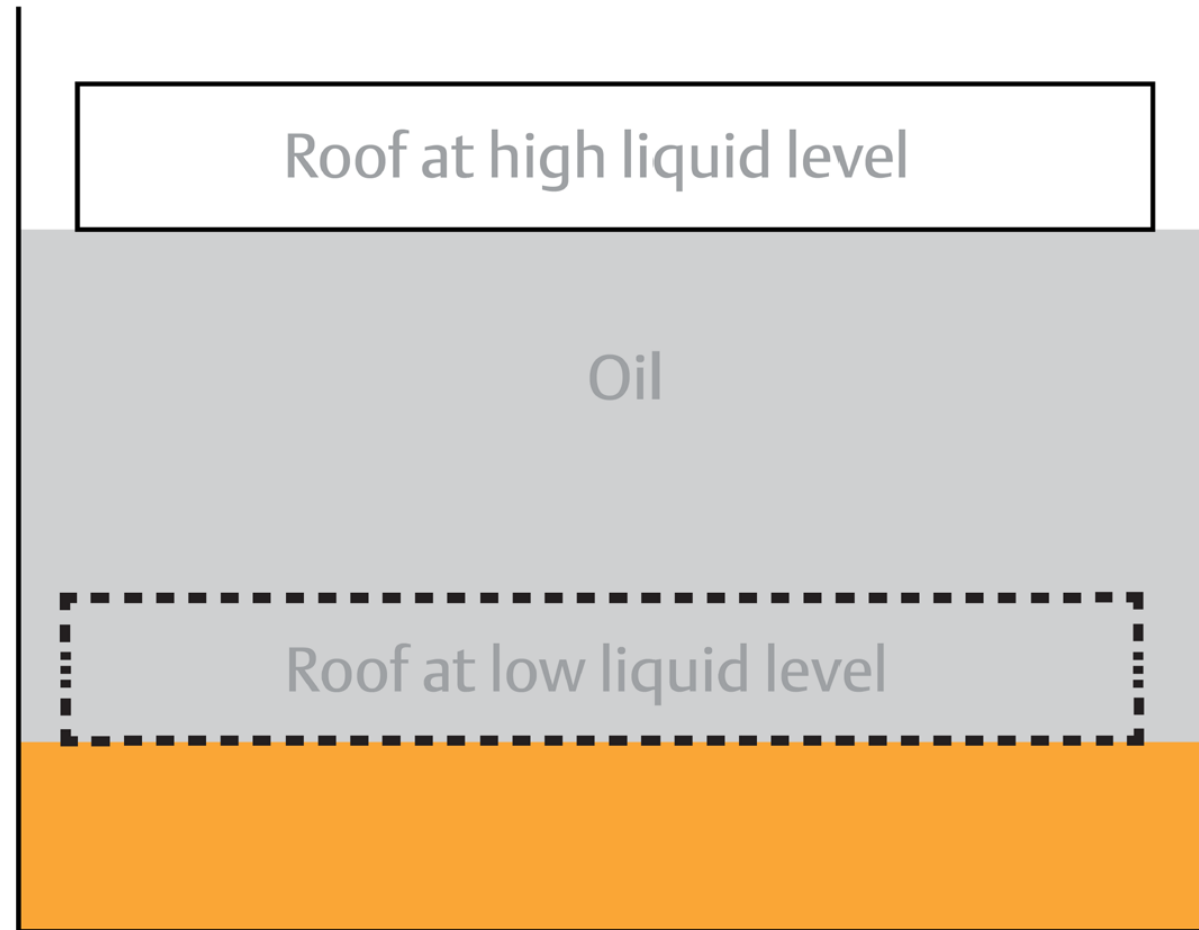


Source: OGJ, July 10, 1989, p. 90.

OGJ

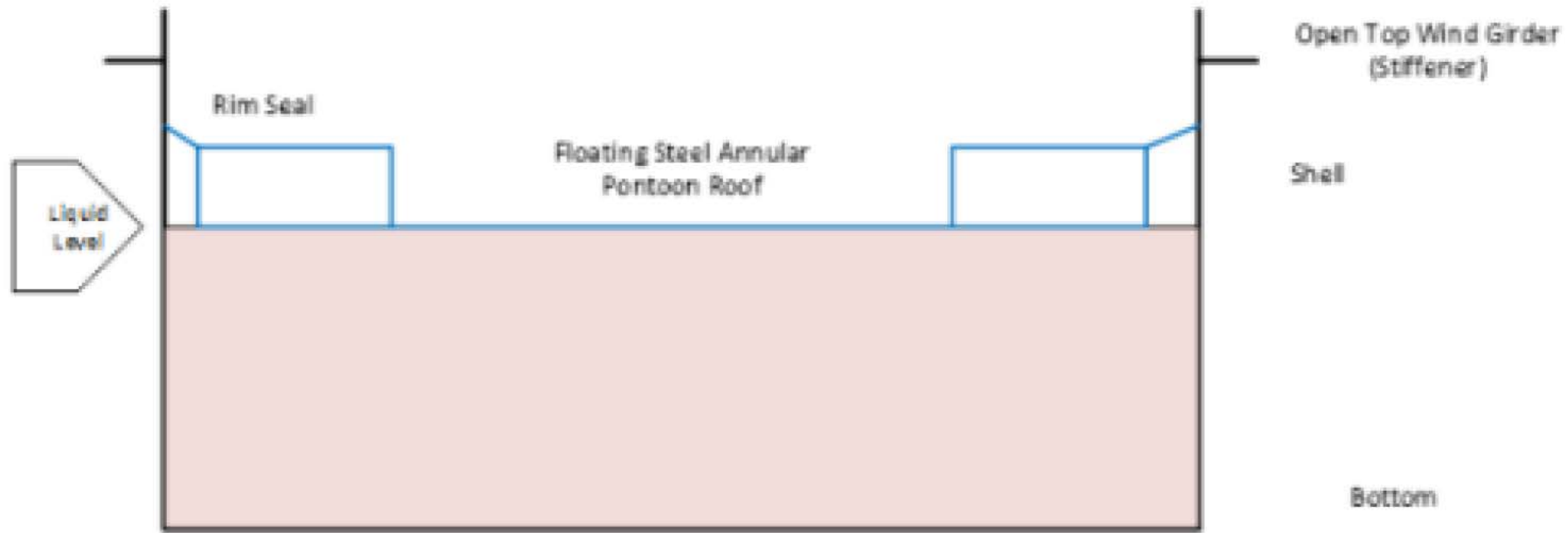
Figure 4 Contrast between geodesic and cone roofs

## Evaporation Reduced

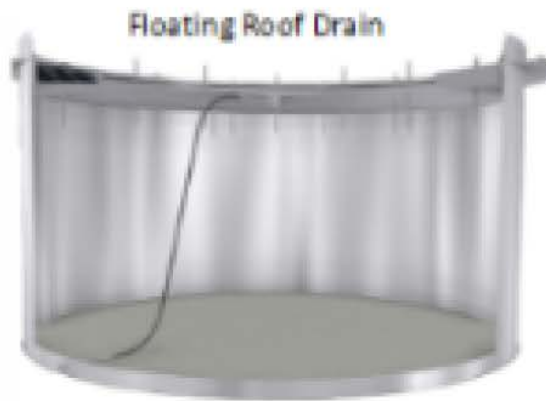


*Figure 1 Conceptual diagram of floating roof tank*

# External or Open Top Floating Roof Tank



Floating Roof Near Top



Floating Roof Drain



Floating Roof seal

Figure 26 Open Top or External Floating Roof Tank (EFRT)

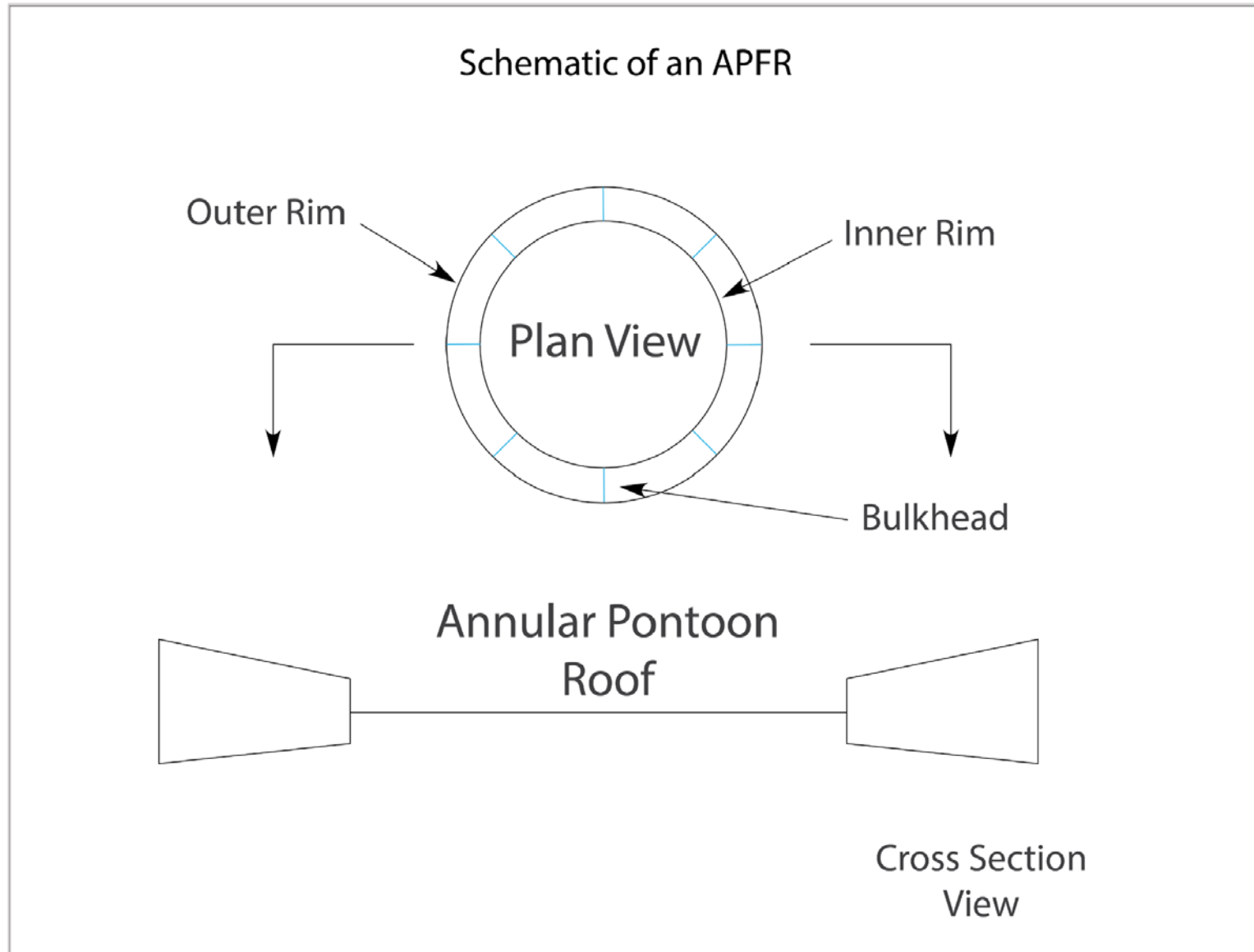
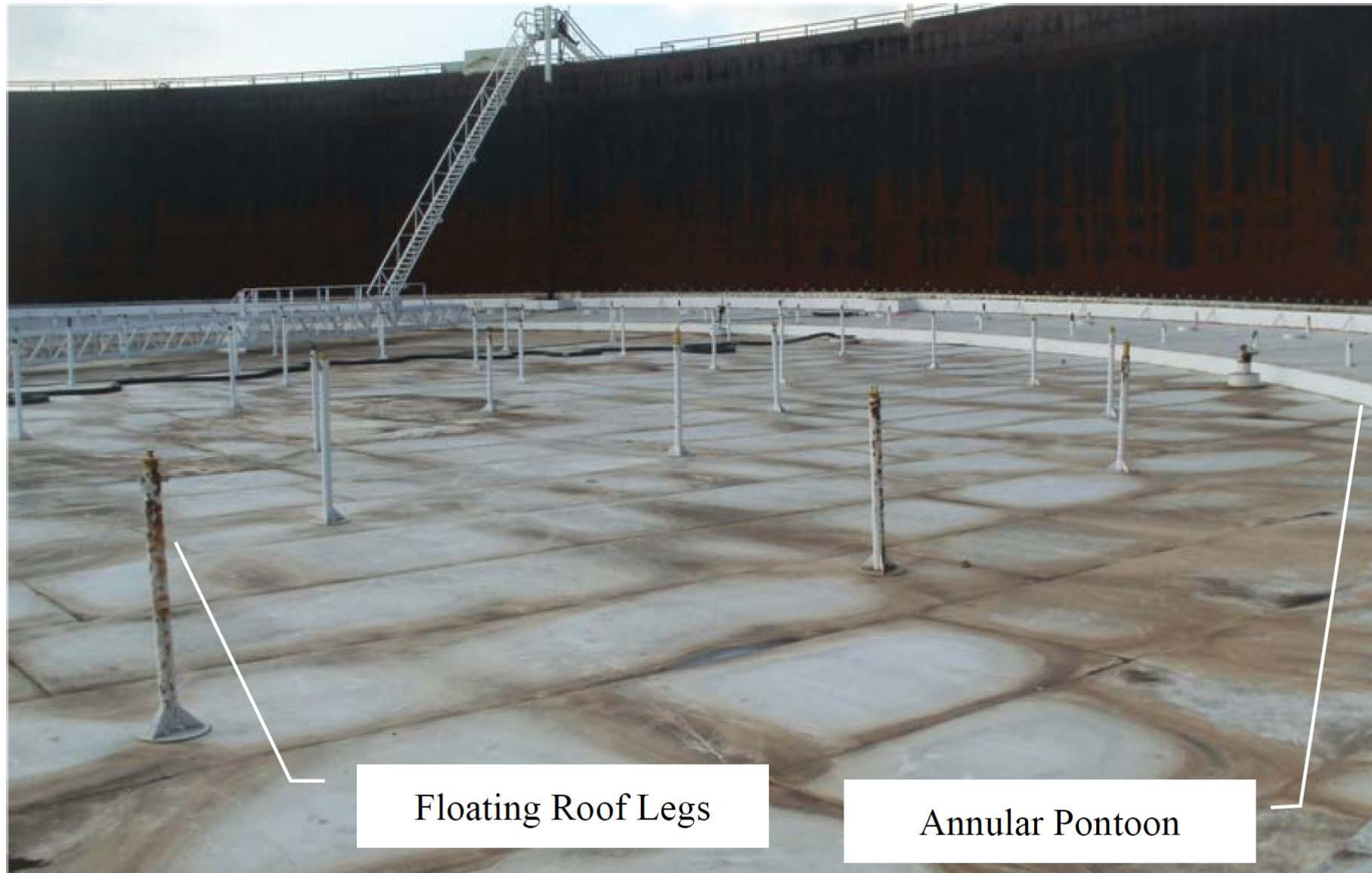
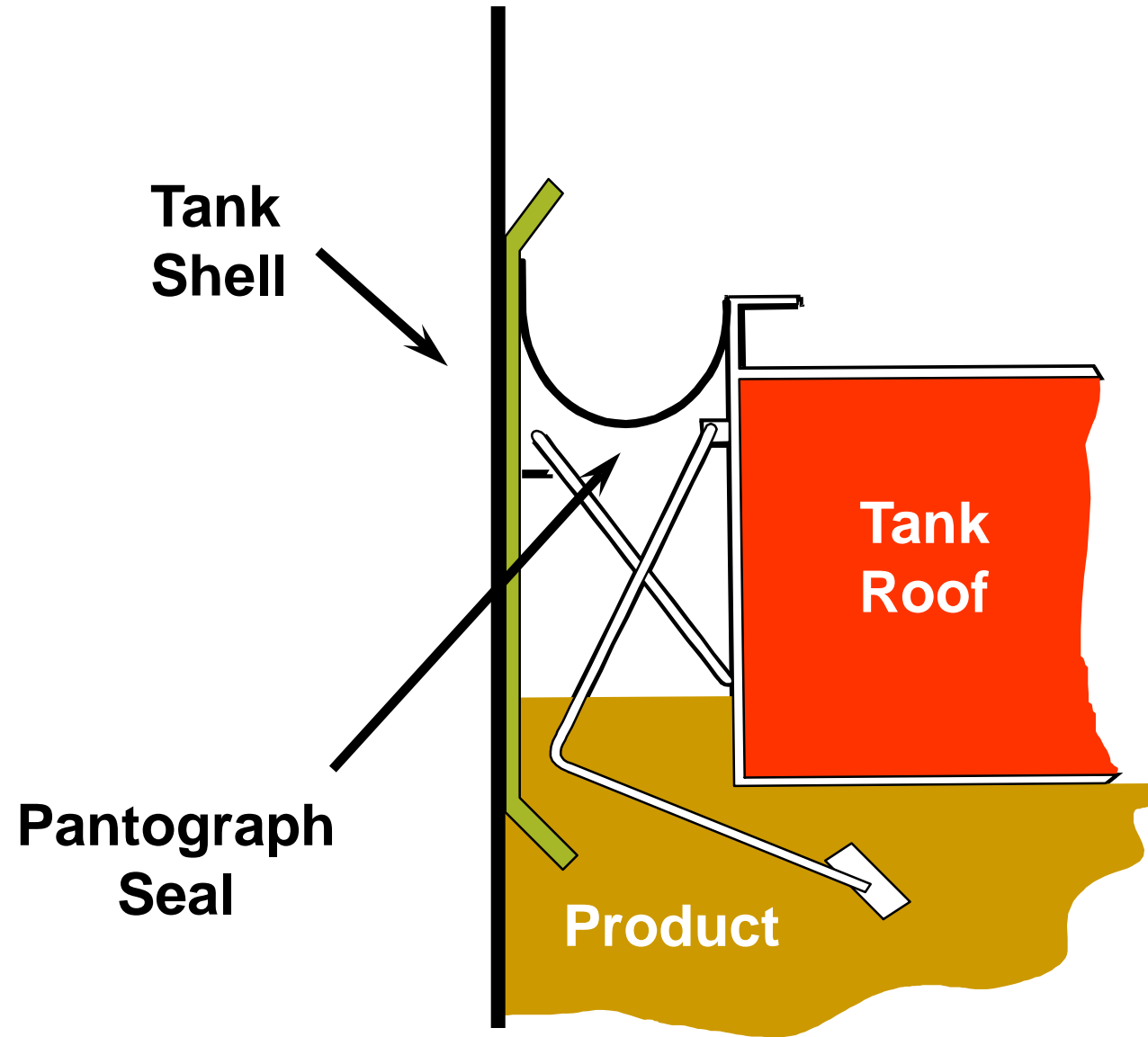


Figure 5 Schematic of an APFR



*Figure 6 An overview of an APFR in application*





## Storage Options for Bulk Liquids

Product	FRT	EFRT	IFRT
Crude Oil (low vapor pressure)	Yes	Yes	Yes
Crude Oil (high vapor pressure)	No*	Yes	Yes
Gasoline	No*	Yes	Yes
Diesel Fuel	Yes	Yes	Yes
Ethanol	No	Yes	Yes

*Table 1 Storage Tanks by Bulk Liquid*

\*In upstream tanks, volatile crude oil is often stored in small tanks (< 30ft diameter). When regulations require or when the owner must control vapors, then vapor recovery systems can be applied to these fixed roof tanks. If storing volatile organic liquids in a fixed roof tank, the pressure-vacuum or PV vent valve should be the vapor recovery system used.



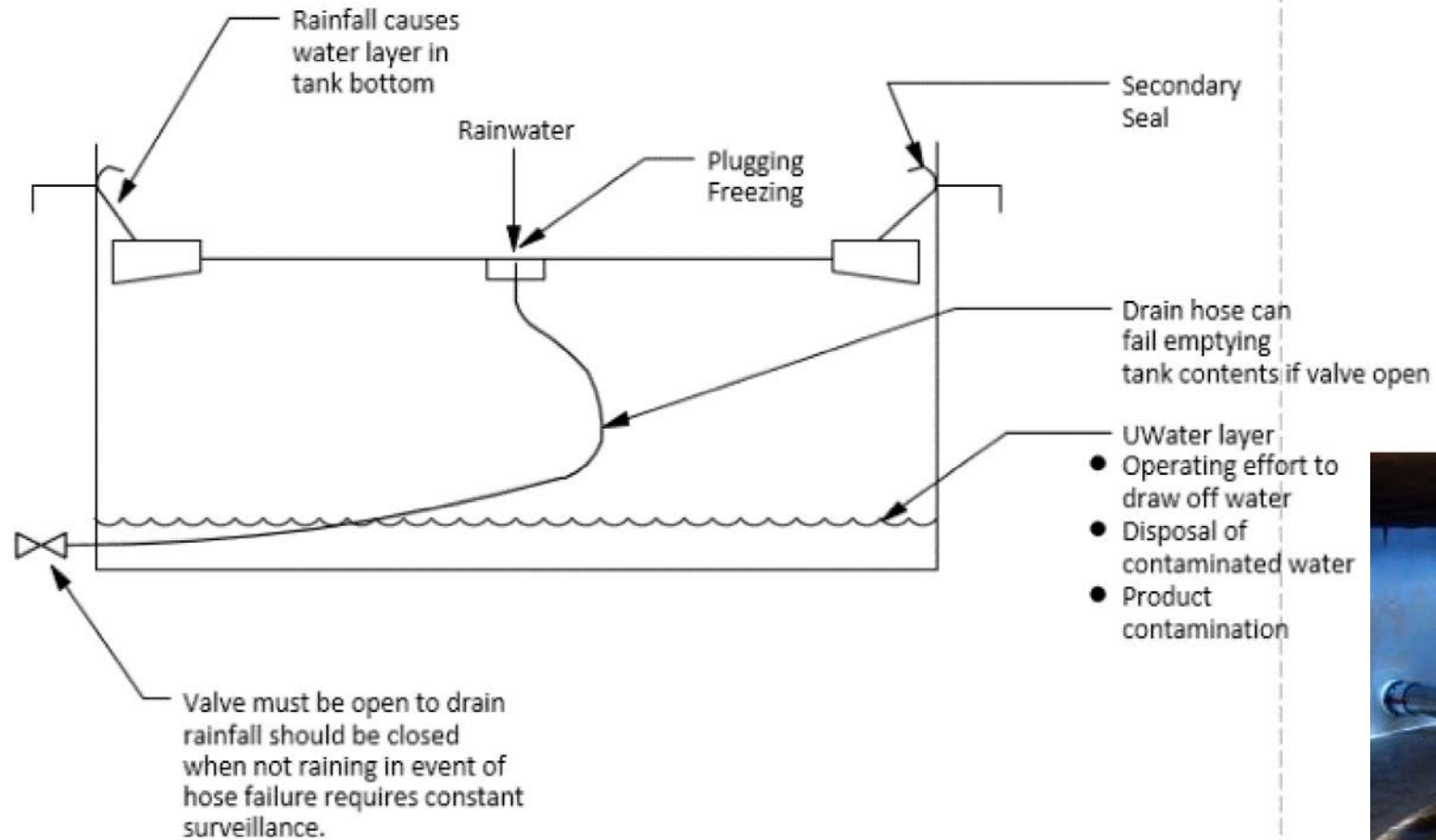


Figure 40 Schematic diagram of APFR drainage systems



*Figure 39 Inadequate precipitation drainage*



# Why API 650?

- Uses
- Specifications
  - 650 v 653
- Types
  - FR, EFR, IFR
- How they differ from small tanks
  - horizontal, vertical, cylindrical, boxlike, etc,
- Moving parts: “foundation”, floating roofs, liquid level, venting devices, etc.
- The hazards: fire/explosion, spills, vapor clouds, air pollution, leaks, toxicity humans, plants, animals
- How to keep jack in the box

# Aboveground v Belowground

- Pros and cons
  - corrosion
  - impact
  - fire
  - environmental
- Fire code spacing requirements

# Where are API 650 Tanks Found

- Petroleum Terminals
- Pipeline Breakout Terminals
- Refineries
- Marine receiving facilities
- Chemical plants
- Bulk plants
- Lubricant Blending and Packaging Facilities
- Asphalt Plants
- Aviation facilities (and airports)
- Oil and Gas production facilities
- etc



# Limits - but all have exceptions

- Diameter constraint not set by API – but by practical limits
  - No lower limit
  - No upper limit
- Thickness
  - 3/16 to 1.5 in
- Materials: mostly mild steel and some higher strength alloys (exceptions)
- Temperature: -20 to 200 (exceptions)
- Shape: vertical cylindrical flat bottom
- Spacing
- Stored liquid vapor pressure

# Constructing Tanks – Which One?

- API 650
- API 653
- API 620
- UL 142
- STI SP001
- ASME BPV

# Materials of construction?

- Mild steel
- Aluminum
- Stainless steel
- Plastic
- FRP
- Concrete
- Wood
- Glass

# Big v Small?

- What factors govern thickness?
  - welding
  - general strength/handling
  - tank size (diameter, height, gas pressure on top of liquid)
  - pressure
- What about stress?



There are two kinds:

- primary
- secondary

# Hoops stress and minimum thickness



shutterstock.com • 1280724148



# Structural Difference Big and Small Tanks

- Show spreadsheet of required thicknesses

# Shape

Check all that are true:

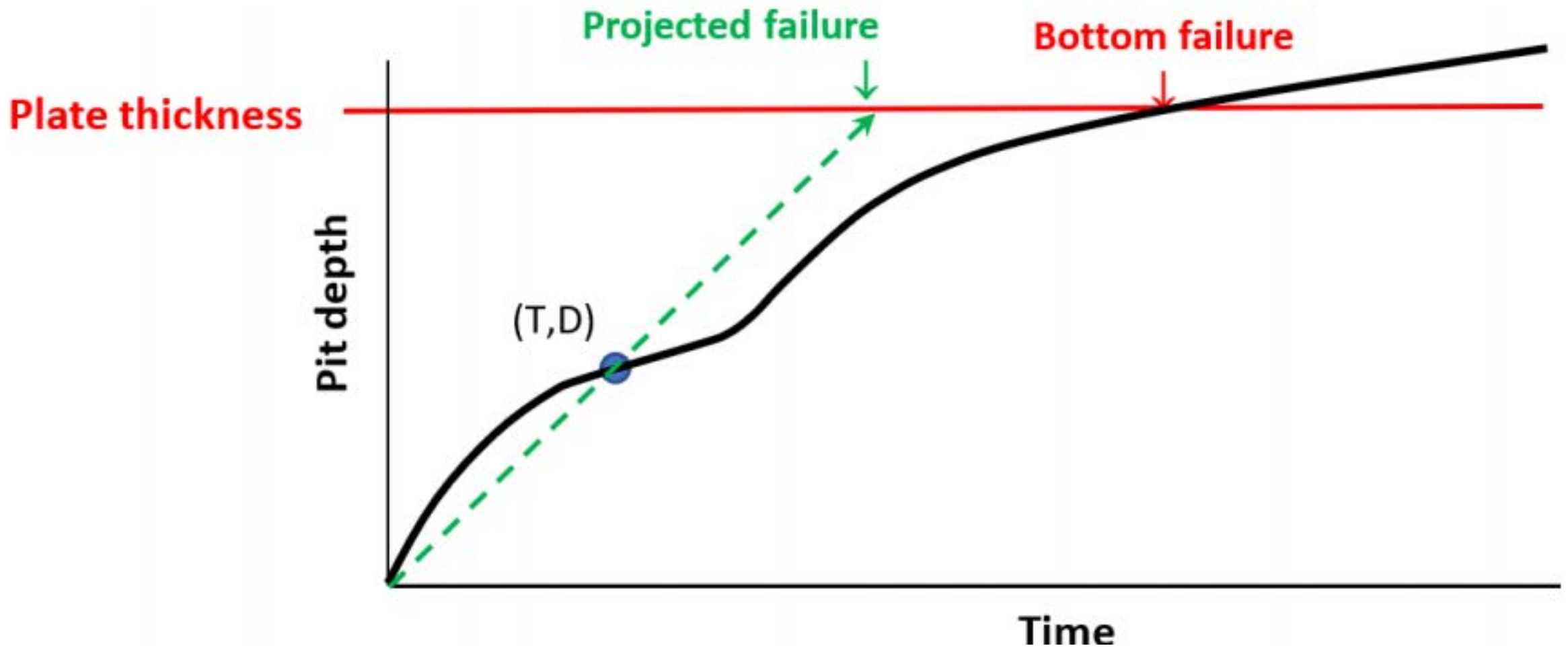
- cylindrical for large tanks
- horizontal for small tanks
- tanks with pressure have flat ends or closures
- vertical tanks may have conical or dished bottoms
- All tanks have fixed covers

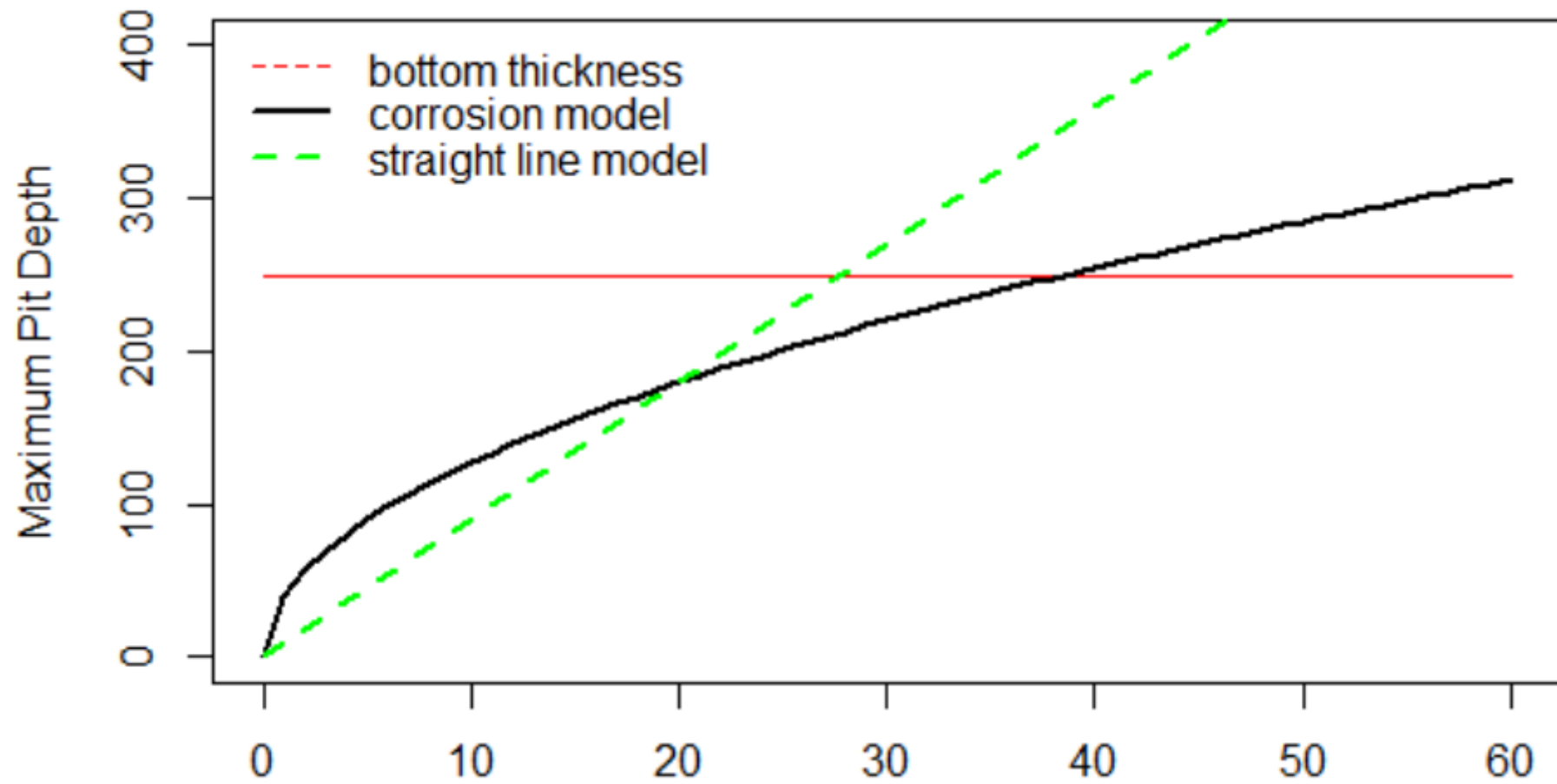


# Then – and Now

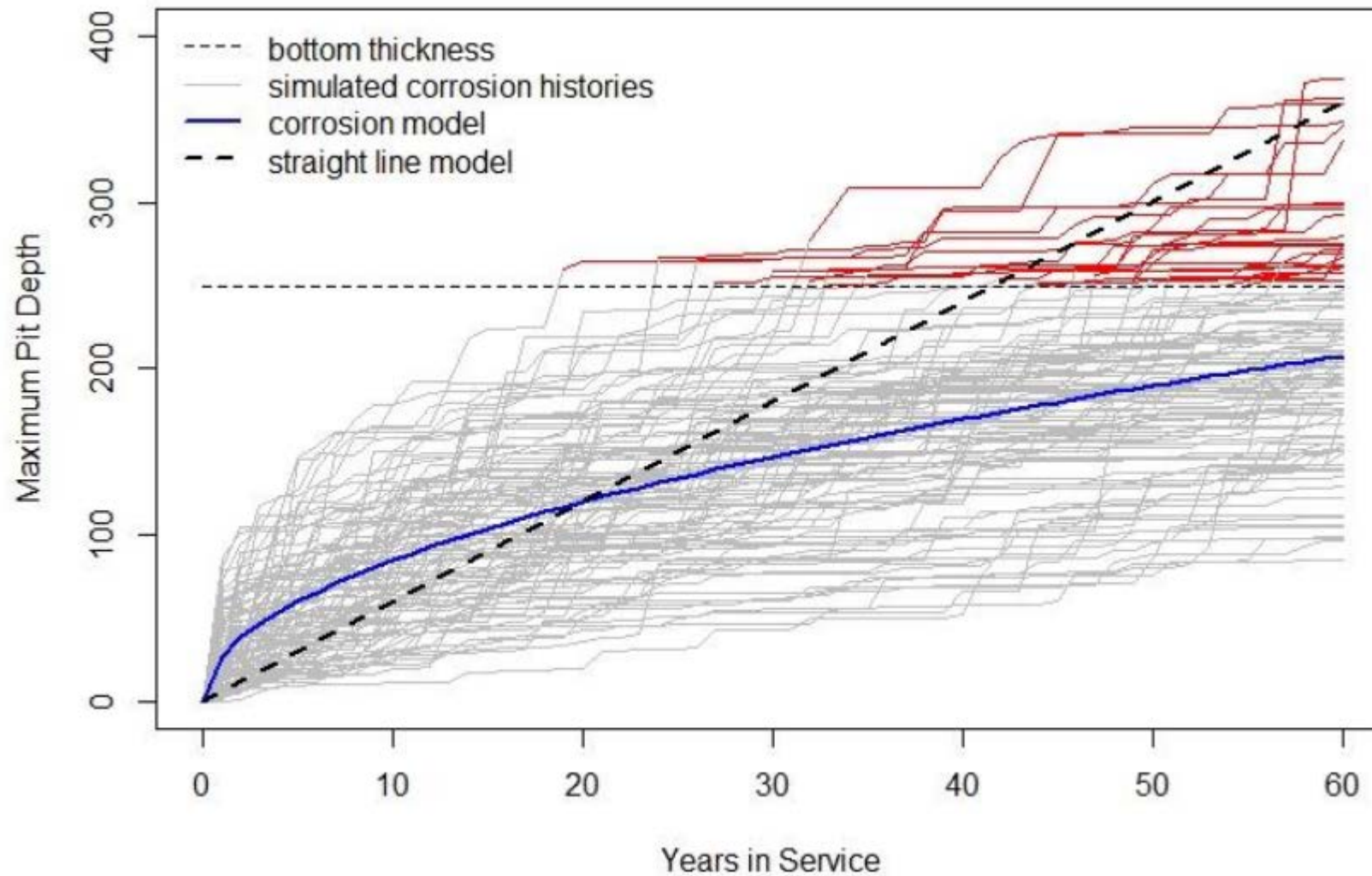
- API 653 issued 1991
- Maximum Inspection interval was 20 yrs
- All tanks should have been inspected by now







# Probabilistic Nature of Corrosion



# Corrosion of Components - Rank

## Vertical Tank

- bottom
- shell
- roof

schematic picture of tank here

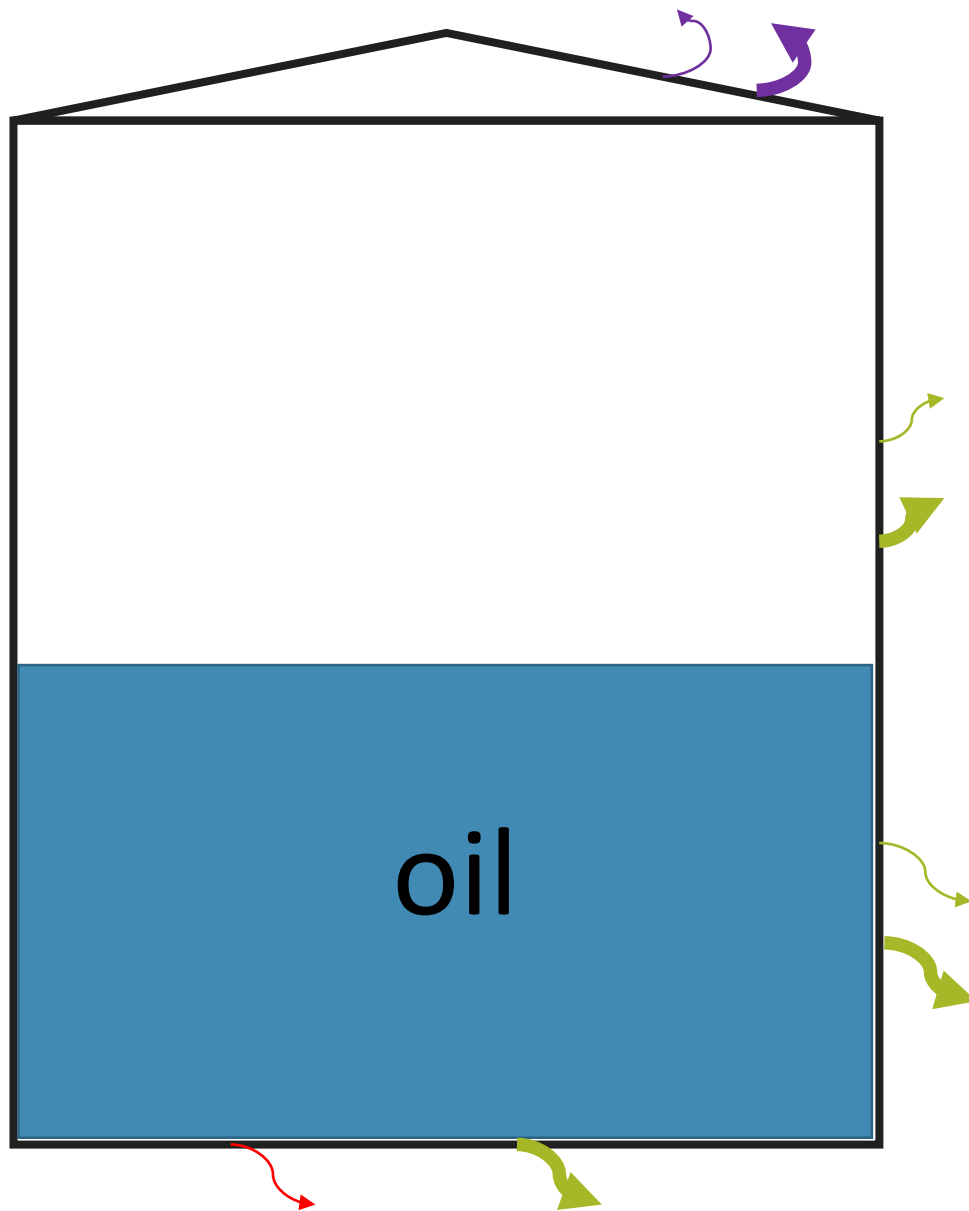
## Horizontal

- bottom
- ends
- shell
- top

# Corrosion – Rank Best to Worst

- aviation gasoline
- heated tank (180F) in refinery service
- lube oils
- gasoline
- crude oil
- diesel

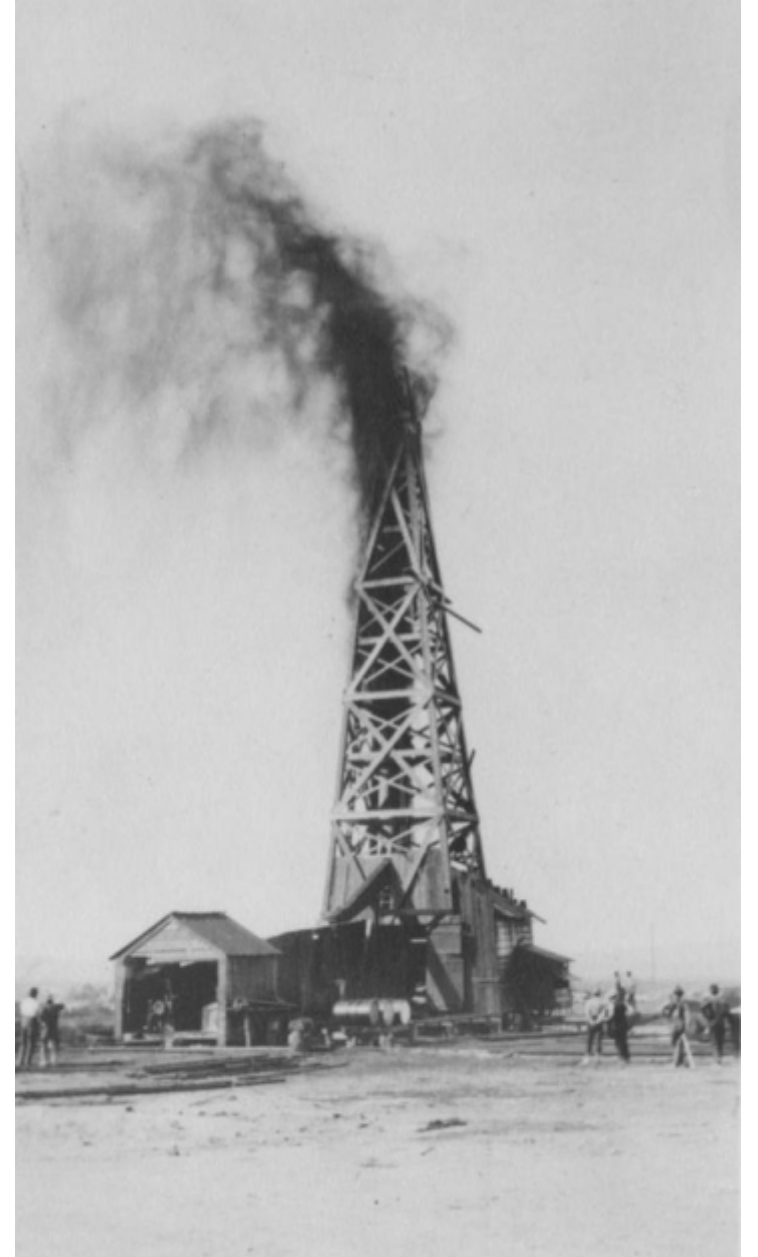
# Leak Causation?



Leaks – which are  
worst?  
best?  
detectable?

# Finding Leaks

- API 653 Monthly walk around
- API 653 Formal External Inspection
- API 653 Formal Internal Inspection

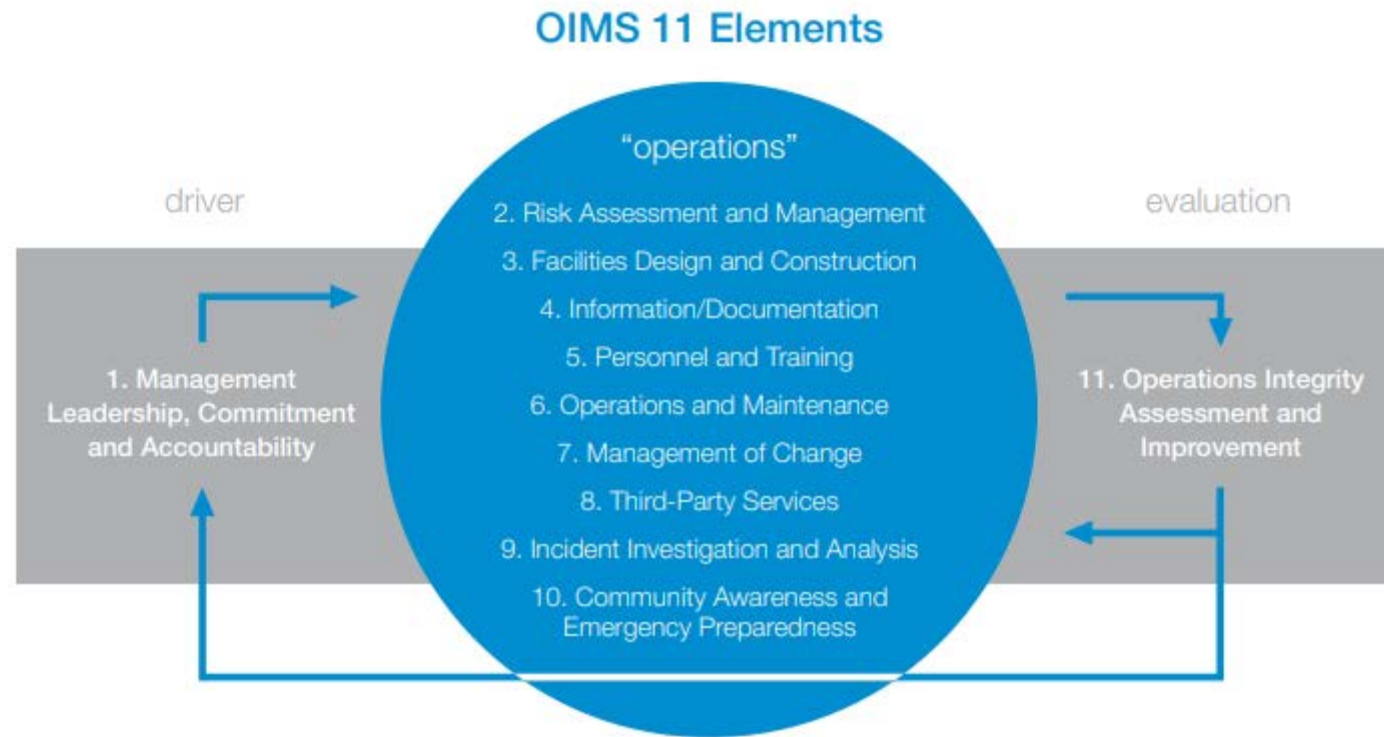




# So What's Really Important for Large ASTs

- API 653 program (verifiable, audiable, and robust)
- Tank overfill protection program API 2350

# How to Keep Jack in the Box



# Mini-Mgmt System for Tanks

- Use industry standards for design, construction, operation
  - API 650
  - API 2610
- Use industry best practices for inspection, repairs, maintenance
  - API 653
  - API 579
- Prevent leaks:
  - Track and understand corrosion data (and insist on it from inspection suppliers)
  - Use stock loss methods
  - Minimize internal inspections

# Then – and Now

- Before 1991 – R you kidding me? You wanna inspect a tank?
- Serious incidents in the 70s and 80s resulted in numerous calls for regulations
- The Ashland Incident along with other pushed API to draft API 653
- The maximum Inspection interval was 20 yrs
- All tanks should have been inspected by now
- The next talk by Brock Trotter will cover tank airborne emissions from storage tanks.



# Tank Emissions Calculations

*How to use API EPA AP 42*

***PEMY Consulting, LLC***

Brock A. Trotter – Process Engineer

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# Background

The US EPA requires companies that store, handle, and transport liquid petroleum products must report the emissions associated. The EPA provides calculation methodologies for a variety of scenarios in AP 42.

The EPA published a modeling software for tank emissions on their website but they state specifically, ***“The model will remain on the website to be used at your discretion and at your own risk. We will continue to recommend the use of the equations/algorithms specified in AP-42 Chapter 7 for estimating VOC emissions from storage tanks. The equations specified in AP-42 Chapter 7 (<https://www.epa.gov/ttn/chief/ap42/ch07/index.html>) can be employed with many current spreadsheet/software programs.”***

# Reference Standards

**API MPMS Ch 19.1 – Evaporative Loss from Fixed-Roof Tanks**

**API MPMS Ch 19.2 – Evaporative Loss from Floating-Roof Tanks**

EPA AP 42 (General Document)



## General Concept

$$E = A \times (EF) \times \left(1 - \frac{ER}{100}\right)$$

*E = emissions, A = activity factor, EF = emissions factor,  
ER = overall emissions reduction efficiency %*

Activity factor determined by the physical properties of the liquid/vapor

Emissions factor determine by the piece of equipment through testing



# API MPMS Ch 19.1\_4.1 – Fixed-Roof Tanks – General

Total routine losses from fixed roof tanks are equal to the sum of the standing loss and working

losses:

$$L_T = L_S + L_W \quad (1-1)$$

where:

$L_T$  = total routine losses, lb/yr

$L_S$  = standing losses, lb/yr, see Equation 1-2

$L_W$  = working losses, lb/yr, see Equation 1-35

# PI MPMS Ch 19.1\_4.2 – Standing Losses

## Overground and Underground Tanks

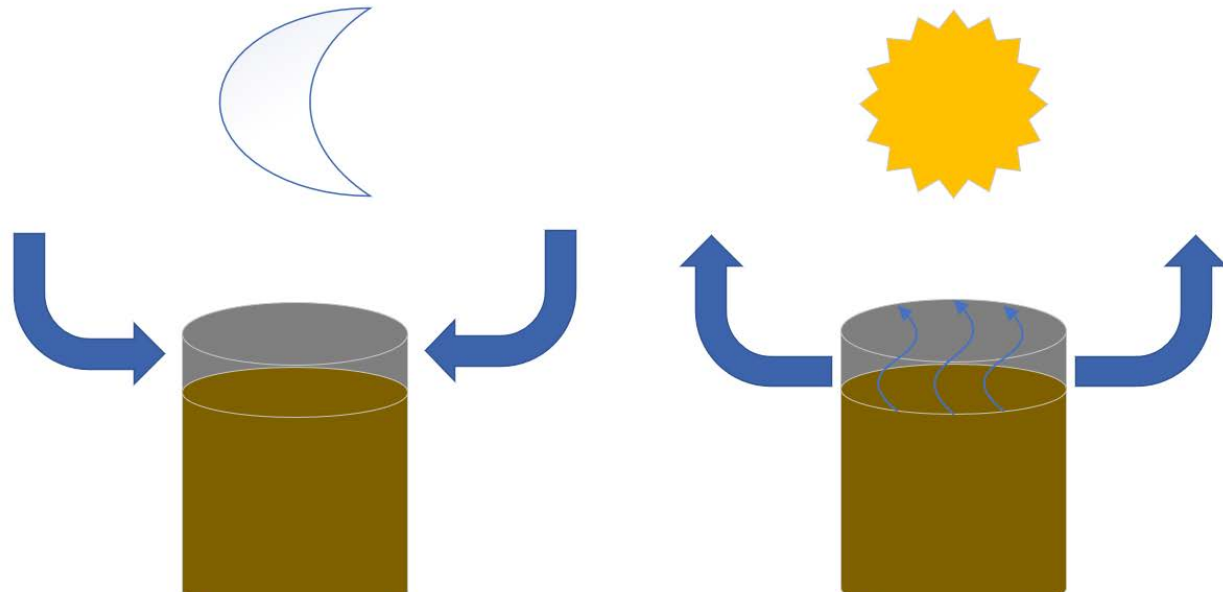
For overground tanks, the standing loss  $L_S$  (lb/yr) is:

$$L_S = 365(\pi D^2/4) H_{VO} K_S K_E W_V \quad (2)$$

where  $H_{VO}$ ,  $K_S$ ,  $K_E$ , and  $W_V$  are determined in 4.2.2 through 4.2.6, respectively;

and the constant 365 has units of days/yr.

For underground tanks, assume no standing loss occurs ( $L_S = 0$ ) because the insulating nature of the earth prevents diurnal temperature change.



# MPMS Ch 19.1\_4.2.3 – Vapor Space Outage Cont.

## Vapor Space Outage $H_{VO}$

Vapor space outage  $H_{VO}$  (ft), the height of a cylinder of diameter  $D$  whose volume equals the vapor volume of a fixed-roof tank, is:

For vertical tanks (see Figure 1):

$$H_{VO} = H_S - H_L + H_{RO}$$

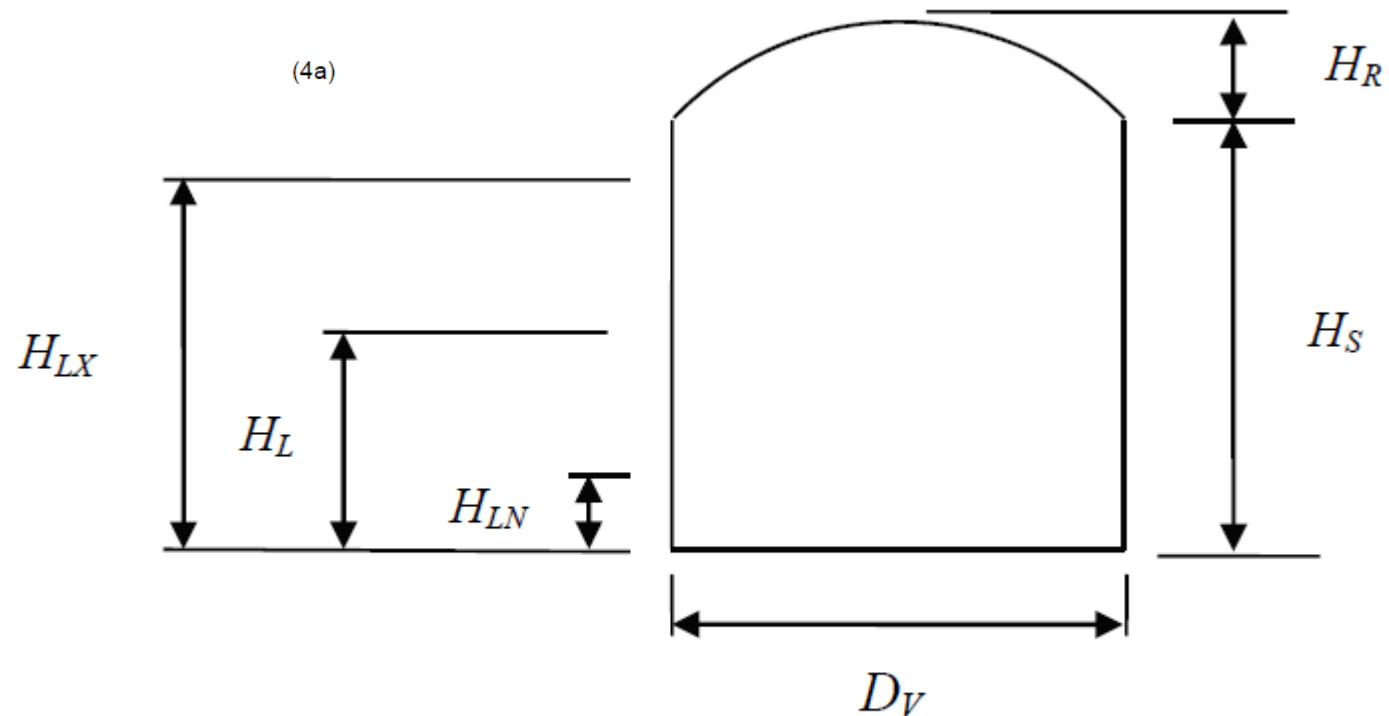


Figure 1 — Fixed-Roof Tank Geometry

# API MPMS Ch 19.1\_4.2.4 – Vented Vapor Sat. Factor

## Vented Vapor Saturation Factor $K_s$

Vapor saturation factor  $K_s$  (dimensionless) accounts for the degree of stock vapor saturation in vapor:

$$1/(1 + 0.053P_{VA} H_{VO}) \quad (7)$$

is determined in 4.2.3;

constant 0.053 has units of 1/(psia-ft).

is the stock true vapor pressure (psia) at the average liquid surface temperature  $T_{LA}$

API MPMS Ch. 19.4, 3<sup>rd</sup> Edition, Section 4.2 to determine vapor pressure  $P_v$  at a given temperature  $T$ )

re

$T_{LA}$  is the daily average liquid surface temperature ( $^{\circ}\text{R}$ ), which may be determined as follows:

$$T_{LA} = 0.44T_{AA} + 0.56T_B + 0.0079aI \quad (8)$$

$$T_{AA} = (T_{AX} + T_{AW})/2 \quad (9)$$

Light – High  $P_{VA}$



Heavy – Low  $P_{VA}$



# API MPMS Ch 19.1\_4.2.4 – Vented Vapor Sat. Factor Cont.

The equation below for estimating liquid bulk temperature is based on the assumption that the product is in thermal equilibrium with ambient air. The time required for the liquid bulk to achieve thermal equilibrium with ambient air, however, would result in the stock typically not being in thermal equilibrium for much of the period. Therefore, it is highly preferable to use measured values for the liquid bulk temperature. If measured values are unavailable,  $T_B$  may be estimated as:

$$T_B = T_{AA} + (6\alpha - 1) \quad (12)$$

where

$\alpha$  is the tank surface solar absorptance (see API MPMS Ch. 19.4, 3<sup>rd</sup> Edition, Section 4.8);

$I$  is the daily total insolation on a horizontal surface (Btu/(ft<sup>2</sup> day)) (see API MPMS Ch. 19.4, 3<sup>rd</sup> Edition, Table 1);

The constants 6 and 1 have units of °R.

Lowest  $\alpha$



Highest  $\alpha$



# MPMS Ch 19.1\_4.2.5 – Vapor Space Expansion Factor

## 4.2.5 Vapor Space Expansion Factor $K_E$

The vapor space expansion factor  $K_E$  is nominally dimensionless but is assigned units of (1/day) because it describes the expansion of vapors in the vapor space that occurs due to the diurnal temperature cycle, and thus it pertains to a daily event.

a) For stocks with  $P_{VA} \leq 0.1$  psia and  $\Delta P_B \leq 0.063$  psi (see Equation 18), the vapor space expansion factor  $K_E$  (1/day) is approximately:

$$K_E = 0.04 \quad (13a)$$

$K_E$  may be estimated more accurately for this case as follows:

$$K_E = 0.0018\Delta T_V \quad (13b)$$

where

The constant 0.0018 has units of  $1/^\circ\text{R}$ .

$\Delta T_V$  is the daily vapor temperature range ( $^\circ\text{R}$ ), which may be determined as follows:

## MPMS Ch 19.1\_4.2.5 – Vapor Space Expansion Factor Cont.

For stocks with  $P_{VA} > 0.1$  or  $\Delta P_B > 0.063$  psi (see Equation 18):

$$K_E = \frac{\Delta T_V}{T_{LA}} + \frac{\Delta P_V - \Delta P_B}{P_A - P_{VA}} \geq 0 \quad (13c)$$

ere

$P_{VA}$  is determined in 4.2.4;

$\Delta T_V$  is determined in 4.2.5a);

$T_{LA}$  is determined in 4.2.4;

$\Delta P_V - \Delta P_B$  is the daily exceedance (psi) of the vapor space pressure range beyond the vent setting range;

$\Delta P_V$  is the daily stock vapor pressure range (psi), and may be determined using either of the following methods:

# MPMS Ch 19.1\_4.3 – Working Losses

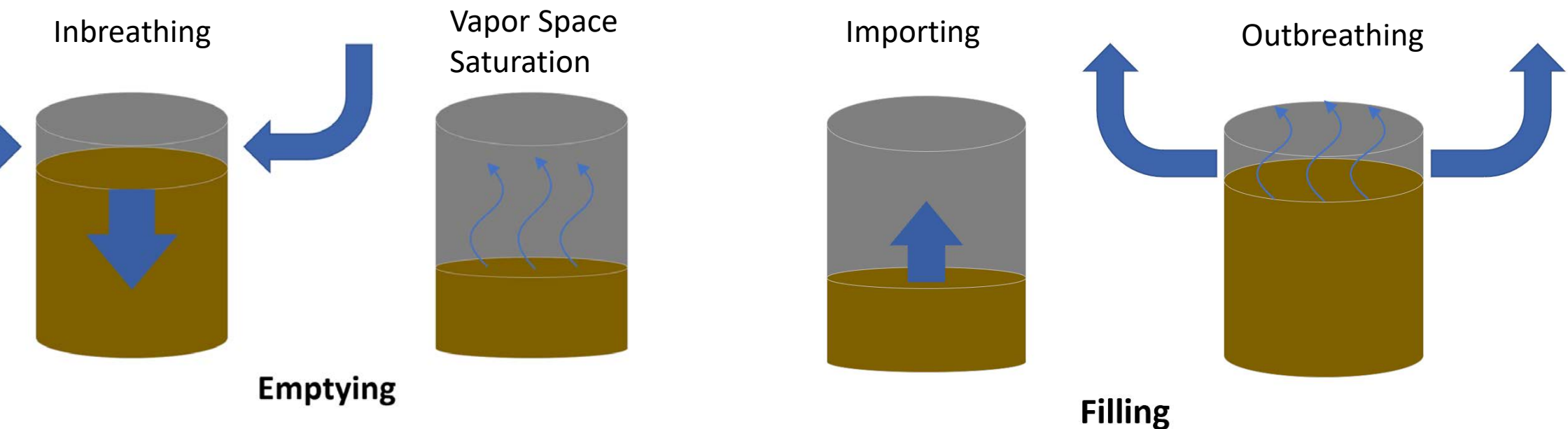
## Working Loss $L_W$

### General

Loss occurs when the liquid level in the tank increases. The working loss  $L_W$  (lb/yr) is:

$$L_W = V_Q K_N K_C K_B W_V \quad (21)$$

$V_Q$ ,  $K_N$ ,  $K_C$ , and  $K_B$  are determined in 4.3.2 through 4.3.5, respectively, and  $W_V$  is determined in 4.2.6.





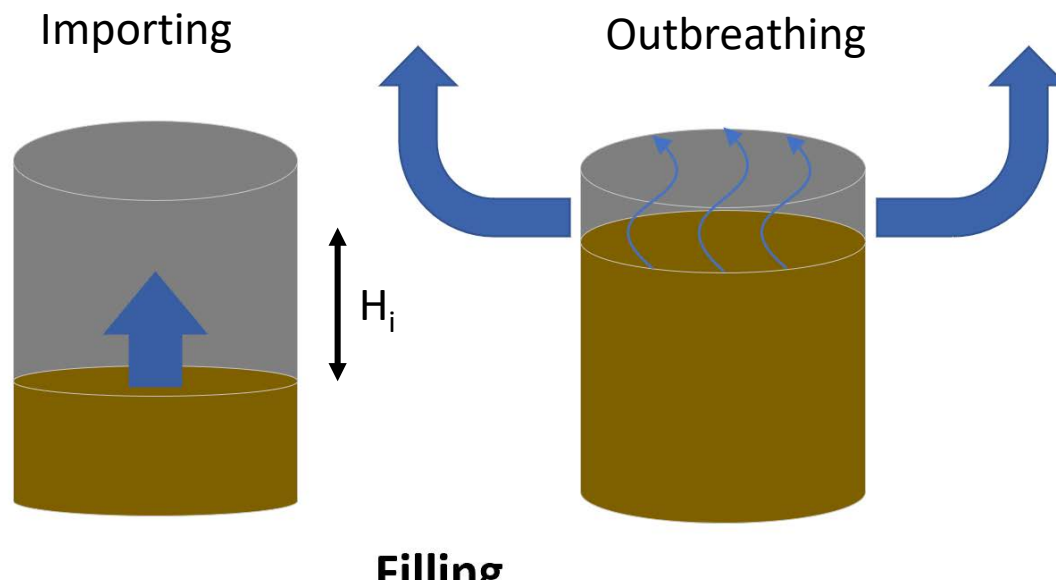
# MPMS Ch 19.1\_4.3.2 – Net Working Loss Throughput

## Net Working Loss Throughput $V_Q$

Working loss throughput ( $\text{ft}^3/\text{yr}$ ) is:

$$V_Q = (\Sigma H_Q)(\pi D^2/4) \quad (22a)$$

$H_Q$  is the annual sum of the increases in liquid level ( $\text{ft}/\text{yr}$ ). If  $\Sigma H_Q$  is unknown,  $V_Q$  can be estimated as:



# MPMS Ch 19.1\_4.3.3 – Turnover Factor $K_N$

## Turnover Factor $K_N$

Turnover factor (dimensionless) is:

$$K_N = 1 \text{ for } N \leq 36 \quad (23a)$$

$$K_N = (180 + N)/(6N) \text{ for } N > 36 \quad (23b)$$

The constant 180 has units of turnovers/yr.

$N$  is the stock turnover rate (turnovers/yr) =

$$\Sigma H_Q / (H_{LY} - H_{LN}) \quad (24a)$$

Dimensionless variable responsible for accounting for the frequency of tank turnovers

# MPMS Ch 19.1\_4.3.4 – Product Factor $K_C$

## 4 Product Factor $K_C$

Product factor accounts for the effect of different stocks on evaporative loss during tank working. The product factor (dimensionless) is:

$$K_C = 0.75 \text{ for crude oil stocks} \quad (26a)$$

$$K_C = 1.0 \text{ for refined petroleum stocks} \quad (26b)$$

$$K_C = 1.0 \text{ for single component petrochemical stocks} \quad (26c)$$

Dimensionless variable responsible for accounting for the product and its respective evaporative loss contribution

# MPMS Ch 19.1\_4.3.5 – Vent Setting Correction Factor $K_B$

## Vent Setting Correction Factor $K_B$

Breather vent pressure setting range  $\Delta P_B$  (determined in 4.2.5b)) is less than or equal to the typical  $\pm 0.03$  psig,  $K_B = 1.0$ . If  $\Delta P_B$  is significantly greater than  $\pm 0.03$  psig:

$$\frac{P_{BX} + P_A}{P_O + P_A} \leq 1.0,$$

$$K_B = 1.0 \quad (27a)$$

$K_N$  is determined in 4.3.3;

$P_A$  is the atmospheric pressure at the tank site (see 4.2.5b2);

$P_{BX}$  is the breather vent maximum pressure setting (see 4.2.5b2);

$P_O$  is the normal operating pressure (psig) =  $(P_{BX} + P_{BN})/2$  (28)

<https://www.youtube.com/watch?v=YA3XUEIcXA0>



## Conclusion Fixed-Roof Tank Emissions

Emissions governed by both thermal fluctuations and liquid level changes

The product determines the physical properties associated with the liquid/vapor equilibrium in the vapor space.

Dark tanks absorb more solar energy and have greater emissions

The EPA does not endorse its prior emissions software and companies must be able to defend values chosen when calculating tank emissions

# MPMS Ch 19.2\_4.1 – Floating-Roof Tanks – General

## General

The total loss  $L_T$  is the sum of the standing loss  $L_S$  and the working loss  $L_W$ :

$$L_T = L_S + L_W \quad (1)$$

Exact same contributions as Fixed-Roof tanks

# MPMS Ch 19.2\_4.2 – Standing Losses

## Standing Loss $L_S$

### Overview

Standing loss pertains to evaporative loss of stock liquid from beneath the floating roof while it is floating. Standing loss  $L_S$  can be estimated as follows:

$$L_S = (F_r + F_f + F_d) P^* M_V K_C \quad (2)$$

- $F_r$  is the total rim-seal loss factor, in pound-moles per year,
- $F_f$  is the total deck-fitting loss factor, in pound-moles per year,
- $F_d$  is the total deck-seam loss factor, in pound-moles per year,
- $P^*$  is the vapor pressure function, dimensionless,
- $M_V$  is the average molecular weight of stock vapor, in pounds per pound-mole,
- $K_C$  is the product factor, dimensionless.

# API MPMS Ch 19.2\_4.2 – Standing Losses Continued

**Table 1—Rim-Seal Loss Factors**

Tank Construction and Rim-seal System	Average-fitting Seals								Tight-fitting <sup>a</sup> Seals							
	Zero-wind Speed Loss Factor	Wind-dependent Loss Factor	Wind-dependent Loss Exponent	Rim-seal Loss Factor $K_r$ (lb-mol/ft-yr)				Zero-wind Speed Loss Factor	Wind-dependent Loss Factor	Wind-dependent Loss Factor	Rim-seal Loss Factor $K_r$ (lb-mol/ft-yr)					
	$K_{r0}$ (lb-mol/ft-yr)	$K_{r0}$ [(lb-mol/(mph) <sup>n</sup> )-ft-yr]	$n$ (dimension-less)	0 (mph)	5 (mph)	10 (mph)	15 (mph)	$K_{r0}$ (lb-mol/ft-yr)	$K_{r0}$ [(lb-mol/(mph) <sup>n</sup> )-ft-yr]	$n$ (dimension-less)	0 (mph)	5 (mph)	10 (mph)	15 (mph)		
<b>Welded Tanks</b>																
<u>Mechanical-shoe seal</u>																
Primary only	5.8 <sup>b,c</sup>	0.3 <sup>c</sup>	2.1 <sup>c</sup>	5.8	15	44	94	1.5	0.4	1.9	1.5	10	33	70		
Shoe-mounted secondary	1.6	0.3	1.6	1.6	5.5	14	24	1.0	0.4	1.5	1.0	5.5	14	24		
Rim-mounted secondary	0.6	0.4	1.0	0.6	2.6	4.6	6.6	0.4	0.4	1.0	0.4	2.4	4.4	6.4		
<u>Liquid-mounted seal</u>																
Primary only	1.6	0.3	1.5	1.6	5.0	11	19	1.0	0.08	1.8	1.0	2.4	6.0	11		
Weather shield	0.7	0.3	1.2	0.7	2.8	5.5	8.4	0.4	0.2	1.3	0.4	2.0	4.4	7.2		
Rim-mounted secondary	0.3	0.6	0.3	0.3	1.3	1.5	1.7	0.2	0.4	0.4	0.2	1.0	1.2	1.4		
<u>Vapor-mounted seal</u>																
Primary only	6.7 <sup>d</sup>	0.2	3.0	6.7	32	210	680	5.6	0.2	2.4	5.6	15	56	139		
Weather shield	3.3	0.1	3.0	3.3	16	100	340	2.8	0.1	2.3	2.8	6.9	23	54		
Rim-mounted secondary	2.2	0.003	4.3	2.2	5.2	62	340	2.2	0.02	2.6	2.2	3.5	10	25		
<b>Riveted Tanks</b>																
<u>Mechanical-shoe seal</u>																
Primary only	10.8	0.4	2.0	11	21	51	100	e	e	e						
Shoe-mounted secondary	9.2	0.2	1.9	9.2	14	25	44	e	e	e						
Rim-mounted secondary	1.1	0.3	1.5	1.1	4.5	11	19	e	e	e						

**Notes:**

<sup>a</sup> "Tight-fitting" means that the floating roof is maintained with no gaps more than 1/8 in. wide between the rim seal and the tank shell.

<sup>b</sup> When no specific information is available, a welded tank with an average-fitting mechanical-shoe primary seal only can be assumed to represent the most common or typical construction and rim-seal system in use on domed EFRTs.

<sup>c</sup> When no specific information is available, a welded tank with an average-fitting mechanical-shoe primary seal only can be assumed to represent the most common or typical construction and rim-seal system in use on EFRTs.

<sup>d</sup> When no specific information is available, a welded tank with an average-fitting vapor-mounted primary seal only can be assumed to represent the most common or typical construction and rim-seal system in use on IFRTs.

<sup>e</sup> No evaporative-loss information is available for riveted tanks with consistently tight-fitting rim-seal systems.



# PI MPMS Ch 19.2\_4.2 – Working Losses

## Working Loss $L_W$

### Overview

ing, or withdrawal, loss pertains to the evaporation of stock liquid that clings to the tank shell (and any roof support columns) while the stock is withdrawn (i.e. while the liquid level is decreased). The working loss  $L_W$  can be estimated as follows:

$$L_W = \frac{0.943 Q_N C_L W_L}{D} \left( 1 + \frac{N_{fc} D_C}{D} \right) \quad (19)$$

$Q_N$  is the net stock throughput associated with decreasing the liquid level in the tank (bbl/yr), - Throughput

$C_L$  is the clingage factor (bbl/1000 ft<sup>2</sup>), - Clingage Loss

$W_L$  is the average stock liquid density at 60°F (lb/gal), - Liquid Density at ST

$N_{fc}$  is the number of fixed-roof support columns (dimensionless),  
- Volume Loss through support column

$D_C$  is the effective column diameter (ft),

$D$  is the tank diameter (ft).

# API MPMS Ch 19.2\_4.3.3 – Clingage Factor

## 4.3.3 Clingage Factor $C_L$

The clingage factor  $C_L$  is given in Table 7.

**Table 7—Clingage Factors  $C_L$  for Steel Tanks (bbl/1000 ft<sup>2</sup>)**

Product Stored	Shell Condition		
	light rust	dense rust	gunite lining
gasoline	0.0015	0.0075	0.15
single-component stocks	0.0015	0.0075	0.15
crude oil	0.0060	0.030	0.60



# Technical Conclusion Floating-Roof Tank Emissions

Emissions governed by both thermal fluctuations and liquid level changes

The product determines the physical properties associated with the liquid/vapor equilibrium in the vapor space.

Simpler to calculate than Fixed-roof tank calculations due to Emissions factor charts

The EPA does not endorse its prior emissions software and companies must be able to defend values chosen when calculating tank emissions



## g Picture Conclusions

Depending on regional windiness, covered tanks have significantly less emissions than open top tanks.

More frequent internal inspections to check seals not necessarily a good idea: tank cleaning emissions are huge (i.e. in the order of a year of operating emissions plus solid wastes). Detail seal inspections should be checked when the tank is out of service for internal inspections.

Minimal inspections can be conducted anytime.

The public and the regulatory agencies can and should write letters to API to effect change. This can be done through inquiries on specific standards as well as writing directly to API. They will act on it.

